

Summary : Solution of  $\ddot{\Theta} + \Gamma \dot{\Theta} + \omega_0^2 \Theta = 0$

(0)  $\Gamma = 0$  No damping

$$\Theta(t) = A \cos(\omega_0 t + \alpha)$$

(1)  $\omega_0^2 > \frac{\Gamma^2}{4}$  Underdamped Oscillator

$$\Theta(t) = A e^{-\frac{\Gamma}{2}t} \cos(\omega t + \alpha) \quad \omega = \sqrt{\omega_0^2 - \frac{\Gamma^2}{4}}$$

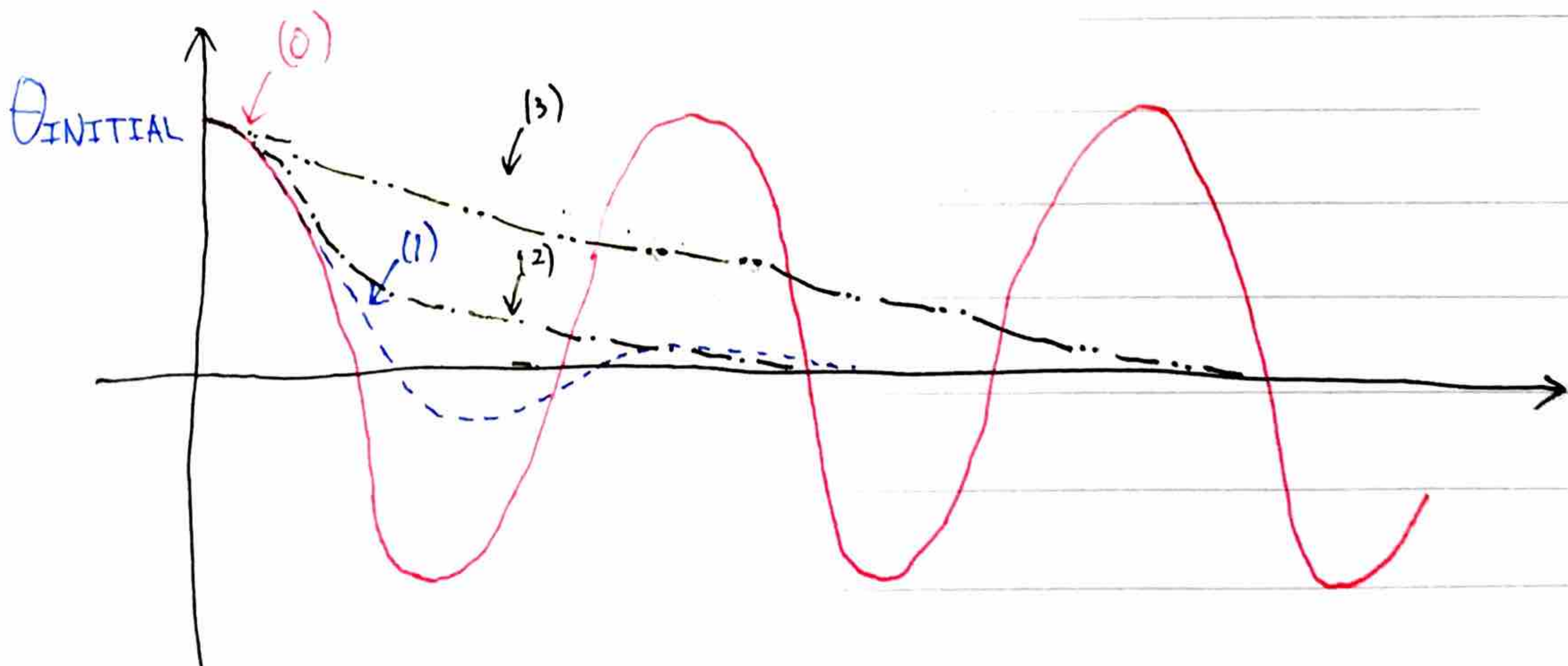
(2)  $\omega_0^2 = \frac{\Gamma^2}{4}$  Critically damped Oscillator

$$\Theta(t) = (A + Bt) e^{-\frac{\Gamma}{2}t}$$

(3)  $\omega_0^2 < \frac{\Gamma^2}{4}$  Overdamped

$$\Theta(t) = A e^{-(\frac{\Gamma}{2} + \beta)t} + B e^{-(\frac{\Gamma}{2} - \beta)t}$$

$$\beta = \sqrt{\frac{\Gamma^2}{4} - \omega_0^2}$$



Slide 5

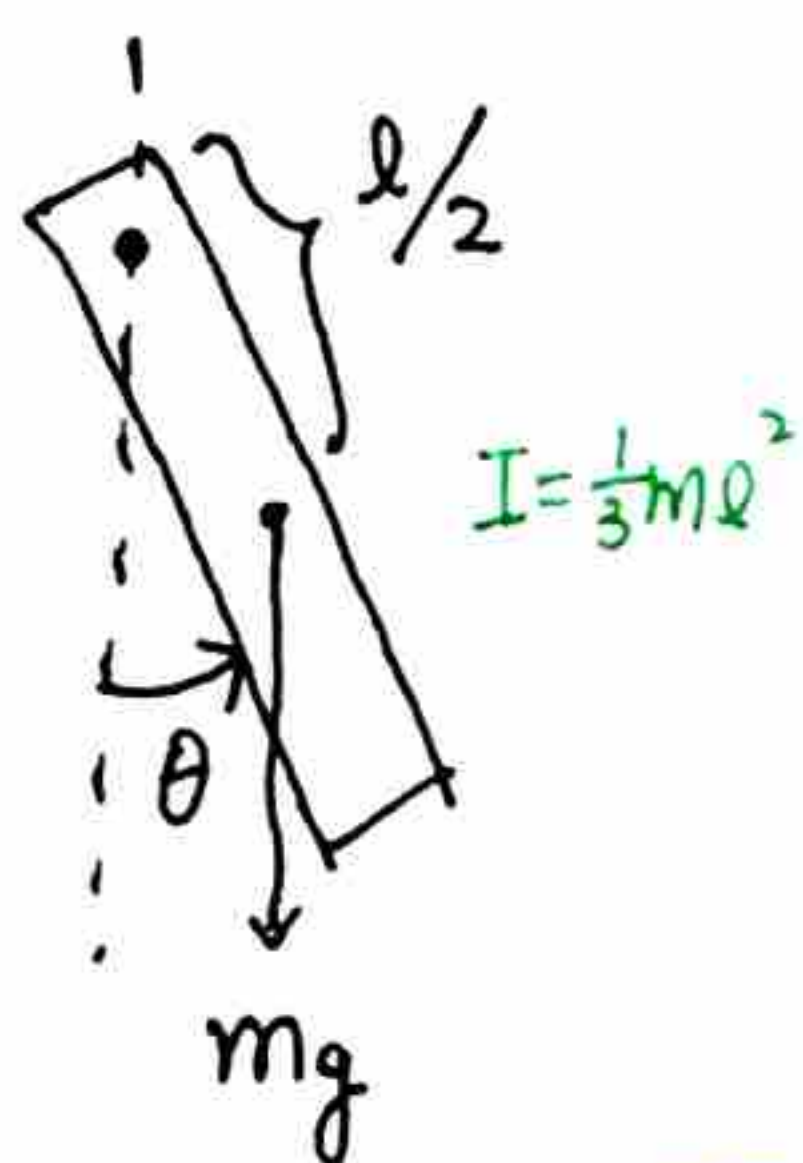


Continue from lecture 2

Now we are interested in giving a driving force to this rod:

Assume that force produce a torque:

$$\tau_{\text{DRIVE}} = d_0 \cos \omega_D t$$



Total torque:

$$\tau(t) = \tau_g(t) + \tau_{\text{DRAG}}(t) + \tau_{\text{DRIVE}}(t)$$

E.O.M:  $\ddot{\theta} + \Gamma \dot{\theta} + \omega_0^2 \theta = \frac{d_0}{I} \cos \omega_D t$

$$\Gamma \equiv \frac{3b}{ml^2}$$



"size of the drag force"

$$\omega_0 \equiv \sqrt{\frac{3g}{2l}}$$



"nature angular frequency"

Define  $f_0 \equiv \frac{d_0}{I}$

$$\Rightarrow \ddot{\theta} + \Gamma \dot{\theta} + \omega_0^2 \theta = f_0 \cos \omega_D t$$

SL7  
Q: oscillation frequency?

We could like to construct something to "cancel"  $\cos \omega_D t$



Idea: use complex notation:

$$\Rightarrow \ddot{z} + \Gamma \dot{z} + \omega_0^2 z = f_0 e^{i\omega_D t}$$



Guess:  $Z(t) = A e^{i(\omega_d t - \delta)}$

Designed to  
cancel  $e^{i\omega_d t}$

It takes some time  
for the system to  
"feel" the driving  
torque.

$$\dot{Z}(t) = i\omega_d Z$$

$$\ddot{Z}(t) = -\omega_d^2 Z$$

Insert those results to the equation of motion:

$$[-\omega_d^2 + i\omega_d \Gamma + \omega_0^2] Z = f_0 e^{i\omega_d t}$$

$$[-\omega_d^2 + i\omega_d \Gamma + \omega_0^2] A e^{i(\omega_d t - \delta)} = f_0 e^{i\omega_d t}$$

⇒ We arrive this expression:

$$[-\omega_d^2 + i\omega_d \Gamma + \omega_0^2] A = f e^{i\delta}$$

$$= f (\cos \delta + i \sin \delta)$$

Since this is a complex equation,

We can solve  $A$  and  $\delta$ :



$$\text{Real Part: } (\omega_0^2 - \omega_d^2) A = J_0 \cos \delta \quad \text{--- (1)}$$

$$\text{Imaginary Part: } \omega_d \Gamma A = J_0 \sin \delta \quad \text{--- (2)}$$

$$(1)^2 + (2)^2: \quad A^2 [(\omega_0^2 - \omega_d^2)^2 + \omega_d^2 \Gamma^2] = J_0^2$$

$$A(\omega_d) = \frac{J_0}{\sqrt{(\omega_0^2 - \omega_d^2)^2 + \omega_d^2 \Gamma^2}}$$

$$\frac{(2)}{(1)}: \quad \tan \delta = \frac{\Gamma \omega_d}{\omega_0^2 - \omega_d^2}$$

$$\Rightarrow \Theta(t) = \text{Re}(Z(t)) = \underbrace{A(\omega_d)}_{\substack{\uparrow \\ \text{Decided by } \omega_d}} \cos(\omega_d t - \underbrace{\delta(\omega_d)}_{\substack{\uparrow \\ \text{Decided by } \omega_d}})$$

No free parameter?! Actually this is particular solution.

The full solution is (if we prepare the system in "under damped" mode)

$$\Theta(t) = \underbrace{A(\omega_d) \cos(\omega_d t - \delta)}_{\substack{\text{Steady State} \\ \text{Solution}}} + B e^{-\frac{\Gamma}{2} t} \cos(\omega t + \alpha)$$

$\omega = \sqrt{\omega_0^2 - \frac{\Gamma^2}{4}}$   
 $\uparrow$

Will die out as  $t \rightarrow \infty$



You may be confused : so many different  $\omega$  ?!

$\omega_0$  : "Natural angular frequency"

In this case:  $\omega_0 = \sqrt{\frac{3g}{2l}}$

$\omega$  : frequency is Lower if there is  
drag force

$$\omega = \sqrt{\omega_0^2 - \frac{\Gamma^2}{4}}$$

$\omega_d$  : frequency of the driving torque  
or force.



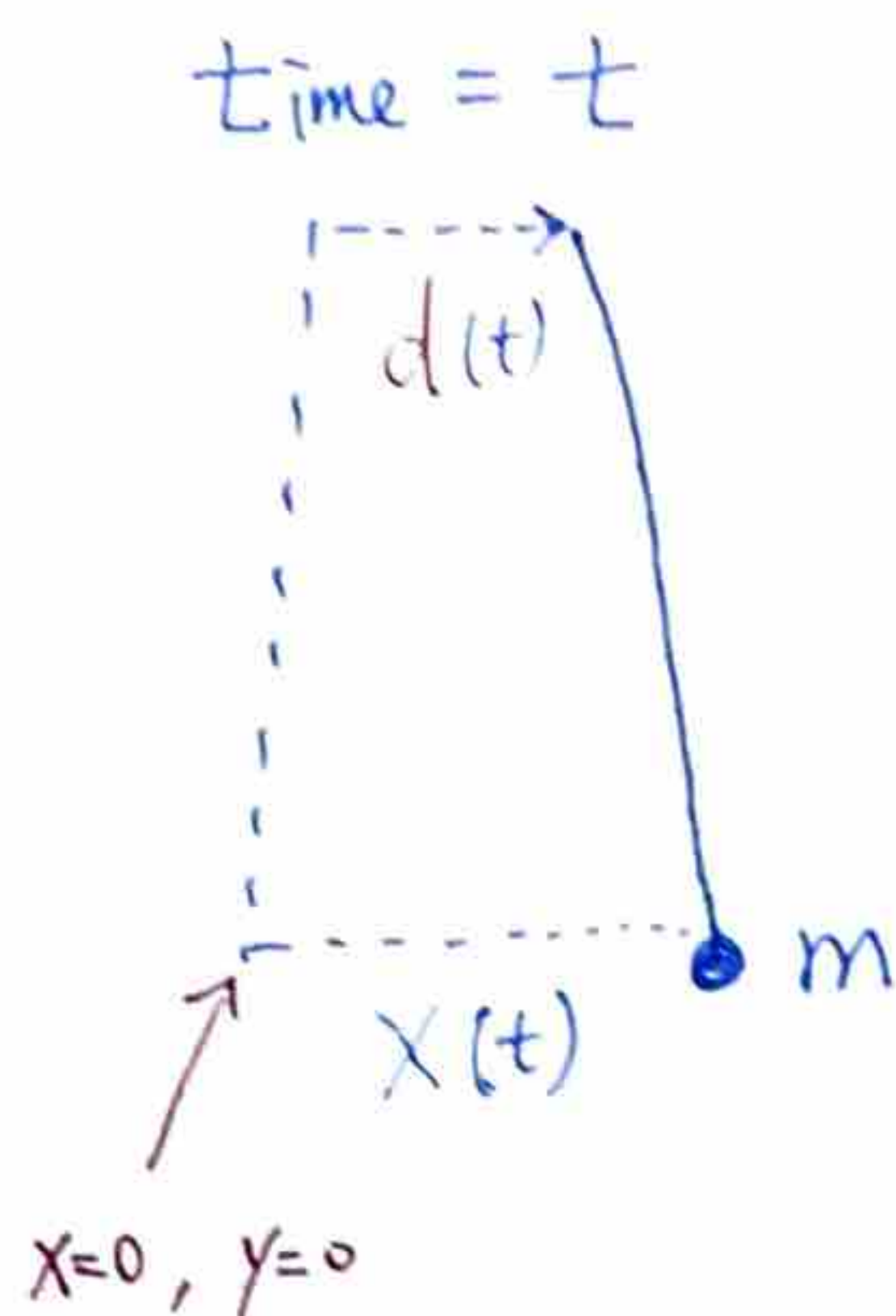
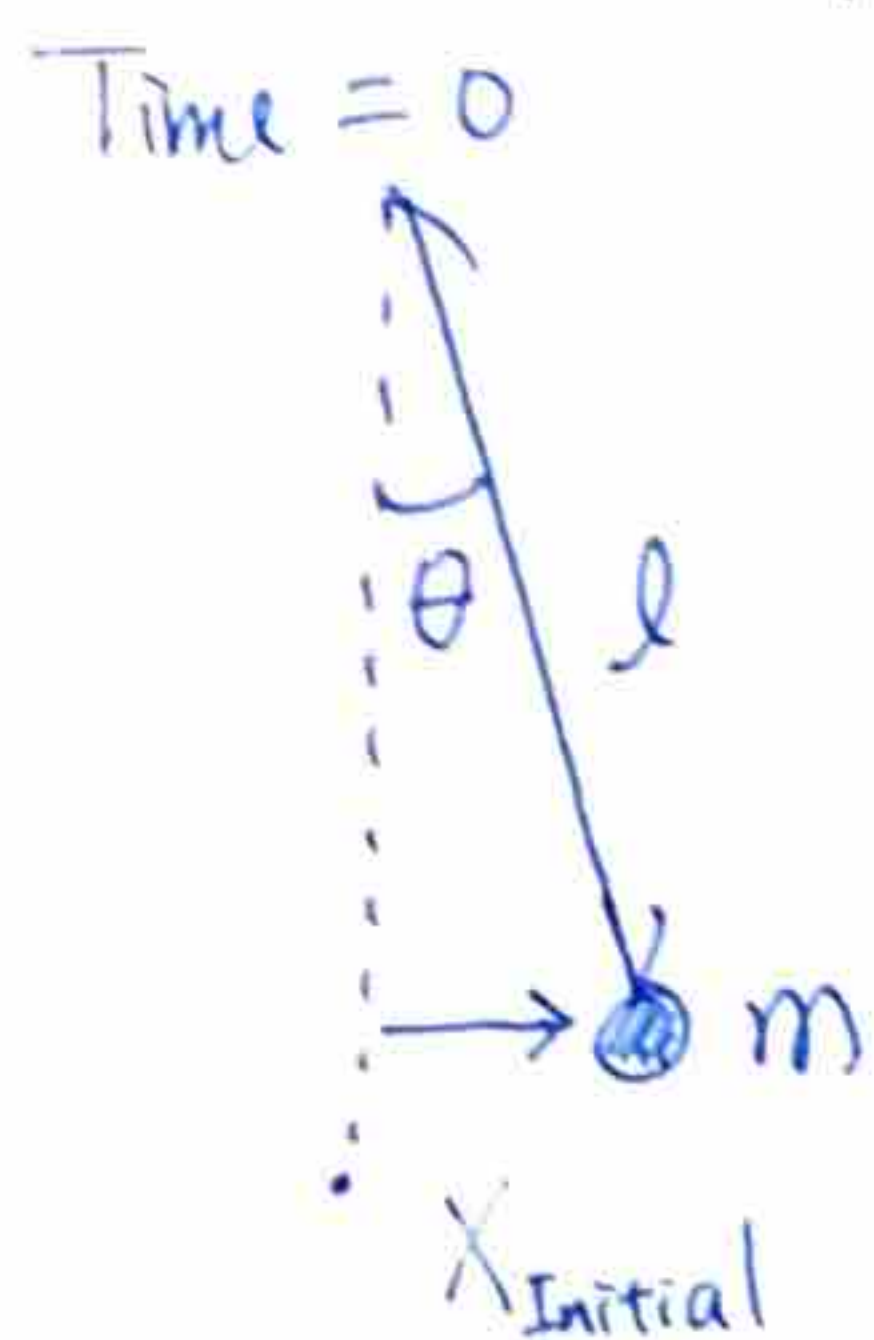
Demo

Cart

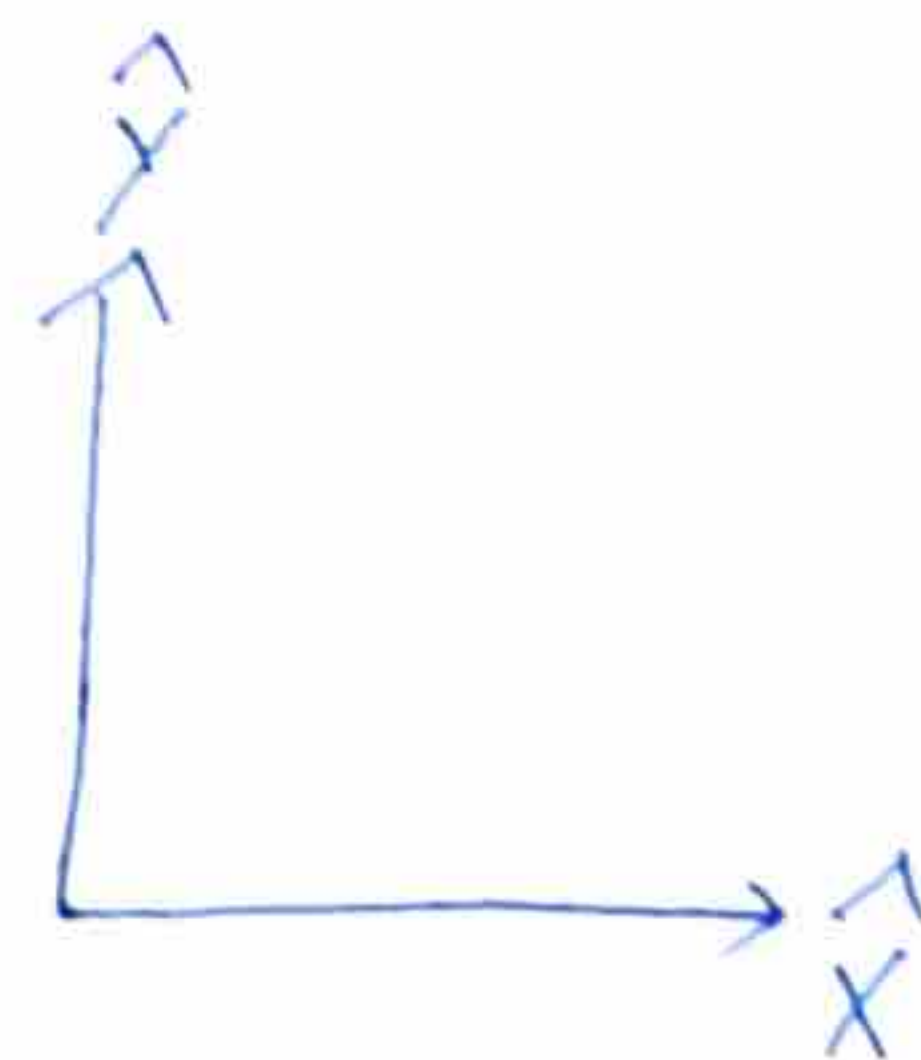
- (1) Natural frequency (Driving force off)
- (2) Demo transient behavior (Driving force on)



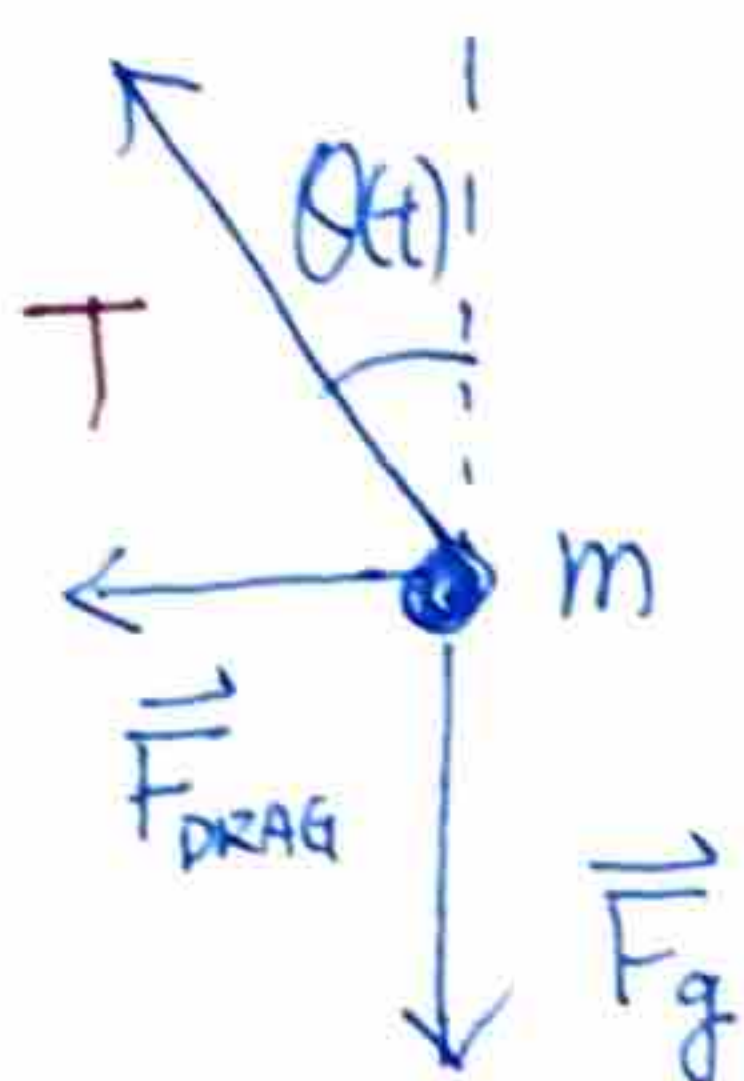
Drive a pendulum:



$$d(t) = \Delta \sin \omega t$$



Force Diagram:



$$\vec{F}_{\text{DRAG}} = -b\dot{x} \hat{x}$$

$$\vec{F}_g = -mg \hat{y}$$

$$\vec{T} = -T \sin \theta \hat{x} + T \cos \theta \hat{y}$$

Take small angle approximation:

$$\sin \theta \approx \theta = \frac{x-d}{l}$$

$$\cos \theta \approx 1$$

$$\Rightarrow \vec{T} \approx -T \frac{(x-d)}{l} \hat{x} + T \hat{y}$$

$$\hat{x}: m\ddot{x} = -b\dot{x} - T \frac{(x-d)}{l}$$

$$\hat{y}: m\ddot{y} = -mg + T$$

|| (small angle)

0

$$\Rightarrow T = mg$$

No vertical motion



$$\Rightarrow m\ddot{x} + b\dot{x} + \frac{mg}{l}x = \frac{mg}{l}d = \frac{mg}{l}\Delta \sin \omega_d t$$

$$\frac{1}{m} \Rightarrow \ddot{x} + \frac{b}{m}\dot{x} + \frac{g}{l}x = \frac{g\Delta}{l} \sin \omega_d t$$

$$\ddot{x} + T\dot{x} + \omega_0^2 x = f_0 \sin \omega_d t$$

$$A(\omega_d) = \frac{f_0}{\sqrt{(\omega_0^2 - \omega_d^2)^2 + \omega_d^2 T^2}}$$

$$(1) \omega_d \rightarrow 0$$

$$\textcircled{1} A(\omega_d) = \frac{f_0}{\omega_0^2} = \frac{\frac{g\Delta}{l}}{g/l} = \Delta$$

The amplitude will be equal to the amplitude  
of Yen-Jie's hand

$\textcircled{2}$

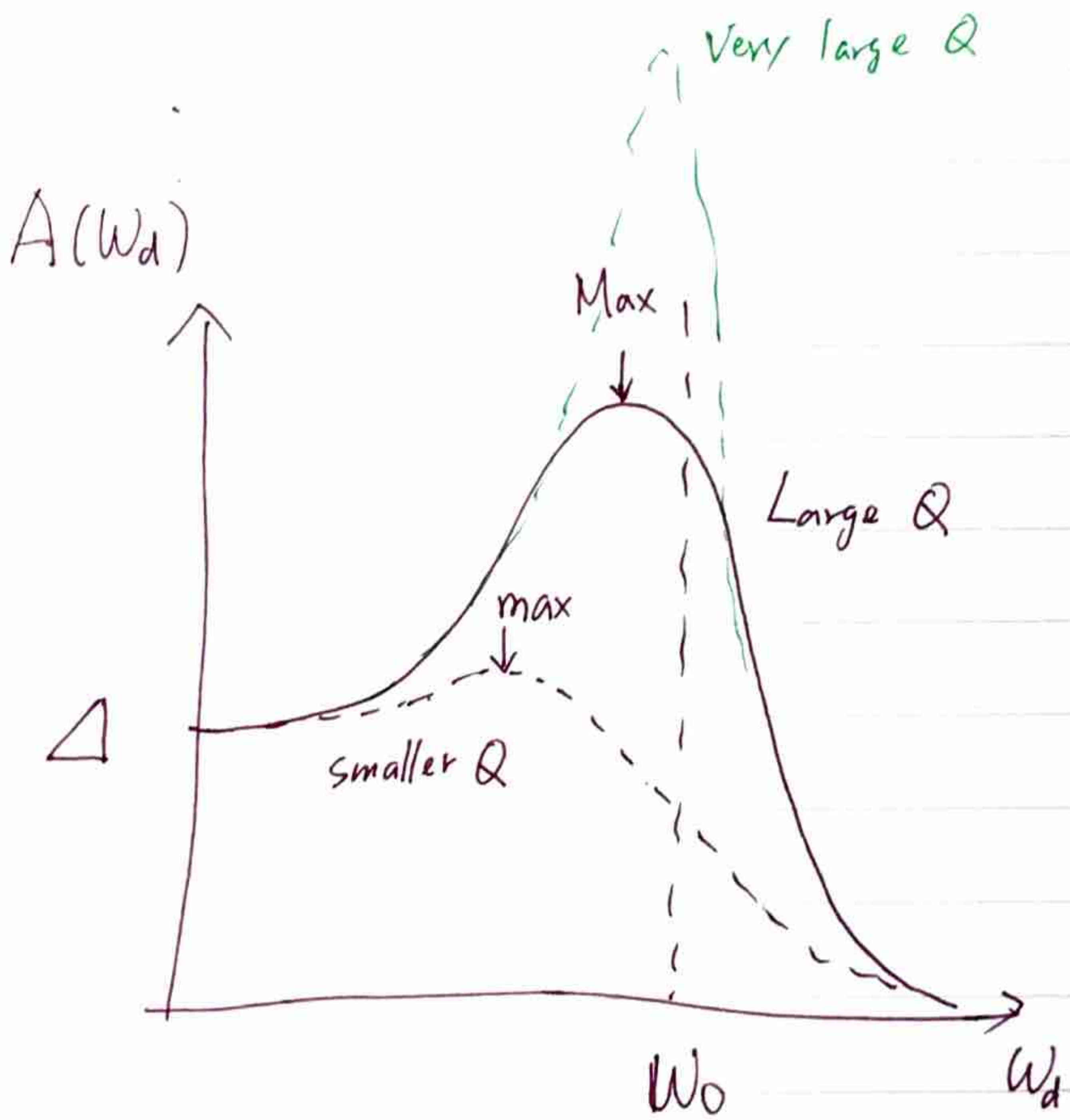
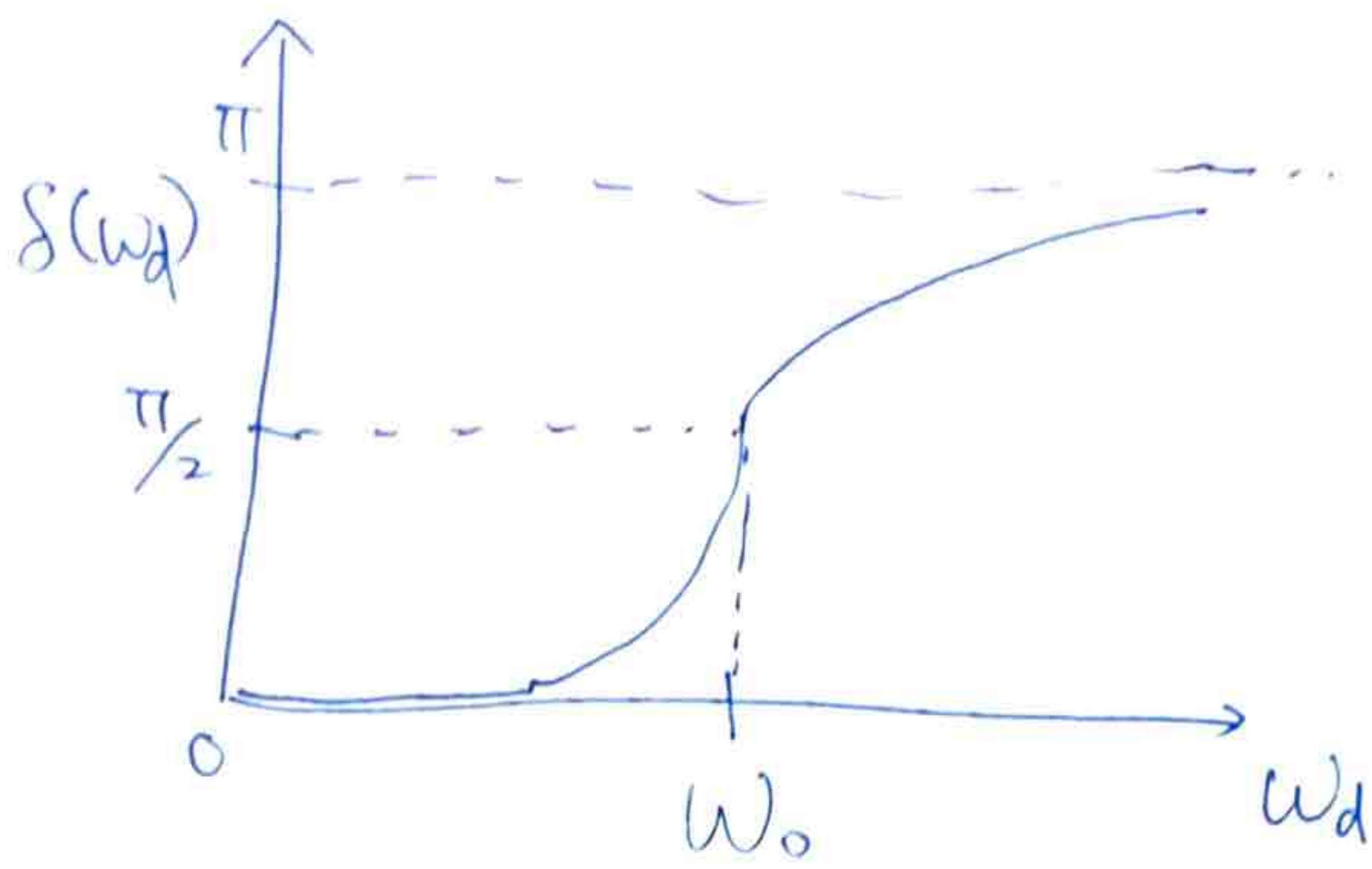
$$\tan \delta = 0 \rightarrow \delta = 0$$

$$(2) \omega_d \rightarrow \infty$$

$$\textcircled{1} A(\omega_d) \Rightarrow 0$$

$$\textcircled{2} \tan \delta \rightarrow \infty \rightarrow \delta = \pi$$





$\omega_{max}$  is slightly smaller than  $\omega_0$



DEMO.



(1) Move extremely slowly

We can see that

$$A(\omega_d) = \Delta$$

$$\delta = 0 \quad (\text{No delay})$$

(2) Move extremely fast

$A(\omega_d)$  very small

$$\delta = \pi$$

(3)  $\omega_d \approx \omega_0$

resonance. Small  $\Delta$  can produce

large  $A$

$$A(\omega_0) = \frac{f_0}{\omega_0 T} = \frac{\omega_0^2 \Delta}{\omega_0 T} = \frac{\omega_0}{T} \Delta = Q \Delta$$

if  $Q \equiv \frac{\omega_0}{T}$  is large

$\Rightarrow$  Large  $A$ .



\* Demo Driven Torsional Balance Oscillator

\* Demo. Driven mechanical oscillator

\* Air condition

\* Washing Machine start to walk around

\* Z bosons (slide 9)

\* Lady breaks the glass by singing

Demo: Break the glass.

15 minutes



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