

2.341 Spring 2014

Problem 7 Solution:

(a) $V_\theta(r, z) = rW(z)$ $V_r = V_z = 0$

$$\underline{\underline{\nabla V}} = \begin{bmatrix} \frac{\partial v_r}{\partial r} & \frac{\partial v_\theta}{\partial r} & \frac{\partial v_z}{\partial r} \\ \frac{1}{r} \frac{\partial v_r}{\partial \theta} - \frac{v_\theta}{r} & \frac{1}{r} \frac{\partial v_\theta}{\partial \theta} + \frac{v_r}{r} & \frac{1}{r} \frac{\partial v_z}{\partial \theta} \\ \frac{\partial v_r}{\partial z} & \frac{\partial v_\theta}{\partial z} & \frac{\partial v_z}{\partial z} \end{bmatrix}$$

$$\Rightarrow \underline{\underline{\nabla V}} = \begin{bmatrix} 0 & W(z) & 0 \\ -W(z) & 0 & 0 \\ 0 & r \frac{dW}{dz} & 0 \end{bmatrix}$$

$$\underline{\underline{\dot{\gamma}}} = \underline{\underline{\nabla V}} + (\underline{\underline{\nabla V}})^T \Rightarrow \underline{\underline{\dot{\gamma}}} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & r \frac{dW}{dz} \\ 0 & r \frac{dW}{dz} & 0 \end{bmatrix}$$

$$\dot{\gamma}_{\theta z} = \dot{\gamma}_{z\theta} = r \frac{dW}{dz}$$

I = $\text{tr}(\underline{\underline{\dot{\gamma}}}) = 0$

II = $\dot{\gamma}_{ij} \dot{\gamma}_{ji} = 0 + \dot{\gamma}_{\theta z} \dot{\gamma}_{z\theta} + \dot{\gamma}_{z\theta} \dot{\gamma}_{\theta z} = 2 r^2 \left(\frac{dW}{dz}\right)^2$

III = $\dot{\gamma}_{ij} \dot{\gamma}_{jk} \dot{\gamma}_{ki} = 0$

But not a homogenous Shear flow

Yes this is a "Shear Flow". ($\dot{\gamma} \neq \text{const.}$)

Each disc at $z = \text{const.}$ is a shearing surface. They move isometrically, and the distance between neighboring points remains const.

(b)

$$\text{Inertia terms} \sim \frac{\rho V^2}{L}$$

$$V \sim R\Omega$$

$$L \sim R$$

$$\text{Viscous terms} \sim \frac{\mu V}{H^2}$$

$$V \sim R\Omega$$

$$\Rightarrow \frac{\text{Inertia}}{\text{Viscous}} \sim \frac{\rho V H}{\mu} \cdot \frac{H}{L} = \frac{\rho(R\Omega)H}{\mu} \cdot \frac{H}{R}$$

$$= \frac{\rho\Omega H^2}{\mu} \ll 1$$

lubrication also needs $\frac{H}{R} \ll 1$

θ -Comp.:

Inertialess:

$$0 = - \left[\frac{1}{r^2} \frac{\partial}{\partial r} (r^2 \tau_{r\theta}) + \frac{1}{r} \frac{\partial}{\partial \theta} \tau_{\theta\theta} + \frac{\partial}{\partial z} \tau_{z\theta} + \frac{\tau_{\theta r} - \tau_{r\theta}}{r} \right] - \frac{1}{r} \frac{\partial p}{\partial \theta} + \rho g_\theta$$

$$\rightarrow \text{Sym. with } \theta \Rightarrow \frac{\partial p}{\partial \theta} = 0, g_\theta \text{ is also } = 0, \frac{\partial}{\partial \theta} = 0$$

also by knowing $\dot{\gamma}_{r\theta} = 0, \dot{\gamma}_{\theta r} = 0 \rightarrow$ the corresponding

stress components

will be zero $\Rightarrow \tau_{r\theta} = \tau_{\theta r} = 0$

Even if you think that $\tau_{r\theta}$ may be still non-zero by lubrication:

$$\frac{1}{r^2} \frac{\partial}{\partial r} r^2 \tau_{r\theta} \sim \frac{\tau}{r} \ll \frac{\tau}{H} \sim \frac{\partial}{\partial z} \tau_{z\theta} \Rightarrow \text{All the terms other than } \tau_{z\theta}$$

$$\Rightarrow \frac{\partial}{\partial z} \tau_{z\theta} = 0 \Rightarrow \boxed{\tau_{z\theta} = \text{Const. with } z} \text{ will vanish ...}$$

(2)

$$\tau_{z\theta} = f_{un}(\dot{\gamma}_{z\theta})$$

$$\tau_{z\theta} = \text{const. with } z \Rightarrow \dot{\gamma}_{z\theta} = \text{const. with } z \Rightarrow r \frac{dW}{dz} = C$$

~~W(z) = C_1 z + C_2~~ ~~W(z) = C_1 z + C_2~~

$$\Rightarrow rW(z) = C_1 z + C_2$$

No slip boundary condition

$$z=0 \Rightarrow rW(z)=0 \Rightarrow C_2=0$$

$$v_{\theta}=0$$

$$z=H$$

$$v_{\theta} = r\Omega \Rightarrow rW(z) = r\Omega = \eta H$$

$$\Rightarrow C_1 = \frac{r\Omega}{H}$$

$$\Rightarrow \boxed{v_{\theta}(r, z) = rW(z) = \frac{r\Omega}{H} z} \Rightarrow W(z) = \frac{\Omega}{H} z$$

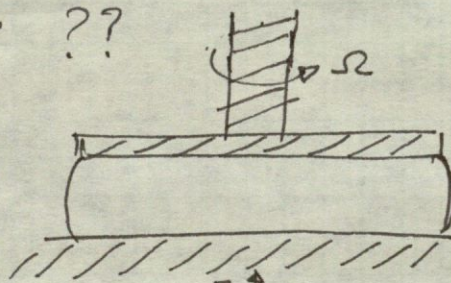
$$\dot{\gamma}_{\theta z} = r \frac{dW}{dz} = r \frac{\Omega}{H} \Rightarrow \boxed{\dot{\gamma}_{\theta z}(r) = \frac{r\Omega}{H}} \text{ changes linearly with } r$$

$$\Rightarrow \dot{\gamma}_{\theta z}(R) = \frac{R\Omega}{H}$$

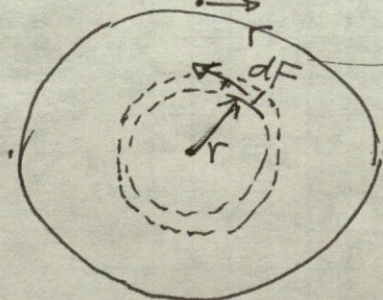
$$\Rightarrow \boxed{\dot{\gamma}(r) = \frac{r}{R} \dot{\gamma}_R}$$

$$(C) T = 2\pi \int_0^R \eta(\dot{\gamma}) \dot{\gamma} r^2 dr \quad ??$$

Side View



Top View



$$dT = r\tau dA$$

$$dF = \tau dA$$

$$\tau = |\tau_{z\theta}| = \eta \dot{\gamma}_{\theta z}$$

$$dA = (2\pi r) dr$$

$$\Rightarrow T = \int dT = \int r dF = \int r \tau dA = \int_0^R r \eta(\dot{\gamma}_{\theta z}) \dot{\gamma}_{\theta z} (2\pi r) dr$$

$$\Rightarrow T = \int_0^R (2\pi) r^2 \eta(\dot{\gamma}) \dot{\gamma} dr$$

It is not possible to explicitly calculate this cause we do not know how $\eta(\dot{\gamma})$ changes with $\dot{\gamma}$ and consequently $r \dots$

$$\dot{\gamma}(r) = \frac{r}{R} \dot{\gamma}_R \Rightarrow dr = \frac{R}{\dot{\gamma}_R} d\dot{\gamma}$$

$$r = \frac{R \dot{\gamma}}{\dot{\gamma}_R}$$

$$\Rightarrow T = \int_0^{\dot{\gamma}_R} (2\pi) \frac{R^2}{\dot{\gamma}_R^2} \dot{\gamma}^2 \eta(\dot{\gamma}) \dot{\gamma} \frac{R}{\dot{\gamma}_R} d\dot{\gamma}$$

$$= 2\pi R^3 \int_0^{\dot{\gamma}_R} \frac{\dot{\gamma}^3}{(\dot{\gamma}_R)^3} \eta(\dot{\gamma}) d\dot{\gamma}$$

$$\Rightarrow T = \left(\frac{2\pi R^3}{\dot{\gamma}_R^3} \right) \int_0^{\dot{\gamma}_R} \eta(\dot{\gamma}) \dot{\gamma}^3 d\dot{\gamma}$$

$$(d) \Rightarrow \frac{T \dot{\gamma}_R^3}{2\pi R^3} = \int_0^{\dot{\gamma}_R} \eta(\dot{\gamma}) \dot{\gamma}^3 d\dot{\gamma}$$

$$\Rightarrow \frac{d(T \dot{\gamma}_R^3 / 2\pi R^3)}{d\dot{\gamma}_R} = \eta(\dot{\gamma}_R) \dot{\gamma}_R^3 \Rightarrow \frac{d(T/2\pi R^3)}{d\dot{\gamma}_R} \dot{\gamma}_R^3 + (T/2\pi R^3) 3\dot{\gamma}_R^2 = \eta(\dot{\gamma}_R) \dot{\gamma}_R^3$$

Leibniz rule

$$\Rightarrow \eta(\dot{\gamma}_R) = \frac{d(T/2\pi R^3)}{d\dot{\gamma}_R} + 3 \frac{(T/2\pi R^3)}{\dot{\gamma}_R}$$

$$\Rightarrow \eta(\dot{\gamma}_R) = \frac{(T/2\pi R^3)}{\dot{\gamma}_R} \left(3 + \frac{d(T/2\pi R^3) / (T/2\pi R^3)}{d\dot{\gamma}_R / \dot{\gamma}_R} \right)$$

(4)

$$\Rightarrow \eta(\dot{\gamma}_R) = \frac{(T/2\pi R^3)}{\dot{\gamma}_R} \left(3 + \frac{d \ln(T/2\pi R^3)}{d \ln \dot{\gamma}_R} \right)$$

$$\Rightarrow \left. \begin{aligned} A &= 2\pi R^3 \\ B &= 3 \end{aligned} \right\}$$

(e) Newtonian: $\eta(\dot{\gamma}) = \mu \Rightarrow T = \int_0^R (2\pi) r^2 \mu \dot{\gamma} dr$ $\dot{\gamma} = \frac{r\Omega}{H}$

$$\Rightarrow T = 2\pi \int_0^R \frac{\mu \Omega}{H} r^3 dr = 2\pi \frac{\mu \Omega}{H} \frac{R^4}{4}$$

$$\Rightarrow \left[T = \frac{2\pi \mu \Omega R^4}{4H} \right]$$

$$T/2\pi R^3 = \frac{\mu \Omega R}{4H}$$

$$\dot{\gamma}_R = \frac{R\Omega}{H} \Rightarrow \frac{T/2\pi R^3}{\dot{\gamma}_R} \left(3 + \frac{d \ln(T/2\pi R^3)}{d \ln \dot{\gamma}_R} \right)$$

$$= \frac{\mu}{4} \left(3 + \frac{d \left[\ln\left(\frac{\mu}{4H}\right) + \ln \dot{\gamma}_R \right]}{d \ln \dot{\gamma}_R} \right)$$

$$= \frac{\mu}{4} (3+1) = \mu \checkmark$$

Power Law: $\eta(\dot{\gamma}) = K \dot{\gamma}^{n-1} \Rightarrow T = \int_0^R 2\pi r^2 K \dot{\gamma}^n dr$ $\dot{\gamma} = \frac{r\Omega}{H}$

$$\Rightarrow T = \int_0^R (2\pi) K \left(\frac{\Omega}{H}\right)^n r^{n+2} dr = \frac{2\pi \Omega^n}{H^n} K \frac{R^{n+3}}{n+3}$$

$$\left. \begin{aligned} T/2\pi R^3 &= \frac{\Omega^n}{H^n} K \frac{R^n}{n+3} \\ \dot{\gamma}_R &= \frac{R\Omega}{H} \end{aligned} \right\} \Rightarrow \dots \text{next page} \dots$$

(5)

$$d \frac{T/(2\pi R^3)}{\dot{\gamma}_R} \left[3 + \frac{d \ln(T/2\pi R^3)}{d \ln \dot{\gamma}_R} \right]$$

$$= \frac{\Omega^{n-1}}{H^{n-1}} K \frac{R^{n-1}}{n+3} \left[3 + \frac{d \left[\ln(K/n+3) + n \ln(\dot{\gamma}_R) \right]}{d \ln \dot{\gamma}_R} \right]$$

$$= \frac{\Omega^{n-1}}{H^{n-1}} K \frac{R^{n-1}}{n+3} (3+n) = K \left(\frac{\Omega R}{H} \right)^{n-1} = K \dot{\gamma}_R^{n-1} = \eta(\dot{\gamma}_R) \checkmark$$

$$(f) \quad \frac{\eta(\dot{\gamma}) - \eta_s}{\eta_0 - \eta_s} = \left[1 + (\lambda \dot{\gamma})^2 \right]^{(n-1)/2}$$

$$\eta_0 = \eta(\dot{\gamma}=0) \rightarrow \approx 2 \text{ Pa}\cdot\text{s}$$

$$\Rightarrow \boxed{\eta_0 \approx 2 \text{ Pa}\cdot\text{s}}$$

$$\eta_s = \eta(\dot{\gamma}=\infty) \Rightarrow \boxed{\eta_s \approx 2-3 \text{ mPa}\cdot\text{s}} \\ \approx 2 \times 10^{-3} \text{ Pa}\cdot\text{s}$$

$\lambda = (\text{Shear rate at which } \eta(\dot{\gamma}) \text{ falling})^{-1}$
Starts

$$= (10^0 \text{ s}^{-1})^{-1} = 1 \text{ s} \Rightarrow \boxed{\lambda \approx 1 \text{ Sec}}$$

As $\eta(\dot{\gamma}) \approx \dot{\gamma}^{n-1} \Rightarrow n-1$ power coefficient
in $\eta(\dot{\gamma})$ curve...

it drops for $10^0 \text{ Pa}\cdot\text{s}$

to $10^{-2} \text{ Pa}\cdot\text{s}$ as we go from 10^0 s^{-1}

$$\text{to } 10^3 \text{ s}^{-1} \Rightarrow n-1 = \frac{-2}{3} \Rightarrow \boxed{n \approx \frac{1}{3}}$$

(6)

(g)

r-Comp.

$$\underline{\underline{\tau}} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & -(\psi_1 + \psi_2) \dot{\gamma}^2 \eta \dot{\gamma} \\ 0 & \eta \dot{\gamma} & -\psi_2 \dot{\gamma}^2 \end{bmatrix}$$

Inertia ignored:

$$\frac{1}{r} \frac{\partial}{\partial r} (r \tau_{rr}) - \frac{1}{r} \frac{\partial}{\partial \theta} \tau_{\theta r} - \frac{\partial}{\partial z} \tau_{zr} + \frac{\tau_{\theta\theta}}{r} - \frac{\partial p}{\partial r} + \rho g_r = 0$$

$$g_r = 0; \tau_{rr} = 0; \frac{\partial}{\partial \theta} = 0; \tau_{zr} = 0;$$

AMPAD

$$\Rightarrow \frac{\tau_{\theta\theta}}{r} - \frac{\partial p}{\partial r} = 0 \Rightarrow \boxed{\frac{\partial p}{\partial r} = \frac{\tau_{\theta\theta}}{r}}$$

$$\Rightarrow \int_r^R \frac{\partial p}{\partial r} dr = \int_r^R \frac{\tau_{\theta\theta}}{r} dr \Rightarrow P(R) - P(r) = \int_r^R \frac{-(\psi_1 + \psi_2) \dot{\gamma}^2}{r} dr$$

$$\Rightarrow P_a - P(r) = \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} \frac{-(\psi_1 + \psi_2) \dot{\gamma}^2}{R \dot{\gamma} / \dot{\gamma}_R} \left(\frac{R}{\dot{\gamma}_R} \right) d\dot{\gamma}$$

$P(r) = P_a$
ignoring surface tension

$$\Rightarrow P(r) - P_a = \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} + (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma}$$

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(h)

$$\tau_{zz} dA = dF \Rightarrow (P(r) + \tau_{zz} - P_a) dA = dF$$

$$\Rightarrow dF = (P(r) - P_a + \tau_{zz}) dA$$

$$\Rightarrow dF|_{r=0} = (P(r=0) - P_a + \psi_2 \dot{\gamma}_{r=0}^2) dA$$

(7)

$$= \left(\int_0^{\dot{\gamma}_R} + (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma} \right) dA \rightarrow \text{It is finite}$$

Normal Force on the plate: F

$$dF = (P(r) - P(a) + -\psi_2 \dot{\gamma}^2) dA$$

$$P(r) - P(a) = \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma}$$

$$\Rightarrow F = \int_0^R \underbrace{(2\pi r dr)}_{dA} \left[-\psi_2 \dot{\gamma}^2 + \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma} \right]$$

Knowing that $r = \frac{R\dot{\gamma}}{\dot{\gamma}_R}$ we can change r into $\dot{\gamma}_r$ and the integral will be:

$$F = \left(\frac{2\pi R^2}{\dot{\gamma}_R^2} \right) \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} \left[-\psi_2 \dot{\gamma}^3 d\dot{\gamma}_r + \dot{\gamma} d\dot{\gamma}_r \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma} \right] \quad (*)$$

Using integration by parts and the fact that

$$\int u dv = uv - \int v du$$

(here it's convenient to say $u = \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma}$

and $dv = \dot{\gamma}_r d\dot{\gamma}_r$

$$\Rightarrow v = \frac{\dot{\gamma}_r^2}{2} \text{ and } du = -(\psi_1(\dot{\gamma}_r) + \psi_2(\dot{\gamma}_r)) \dot{\gamma}_r$$

$$\Rightarrow \int_0^{\dot{\gamma}_R} \dot{\gamma}_r d\dot{\gamma}_r \int_{\dot{\gamma}_r}^{\dot{\gamma}_R} (\psi_1 + \psi_2) \dot{\gamma} d\dot{\gamma} = \int_0^{\dot{\gamma}_R} \frac{\dot{\gamma}_r^3}{2} (\psi_1 + \psi_2) d\dot{\gamma}_r$$

$$\Rightarrow \text{Plugging into } (*) : F = \left(\frac{\pi R^2}{\dot{\gamma}_R^2} \right) \int_0^{\dot{\gamma}_R} \dot{\gamma}_r^3 (\psi_1 - \psi_2) d\dot{\gamma}_r$$

(8)

Now similar to parts "c" and "d" you can rearrange and use Leibniz rule to show that:

$$\Psi_1(\dot{\gamma}_R) - \Psi_2(\dot{\gamma}_R) = \frac{1}{\dot{\gamma}_R^2} \left(\frac{F}{\pi R^2} \right) \left[2 + \frac{d \ln(F/\pi R^2)}{d \ln \dot{\gamma}_R} \right]$$

which is a "nice" expression. It helps us to get rheological measurements out of a "non-homogenous" shear flow for any arbitrary liquid with any possible constitutive equation!!

End of solution

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