

MASSACHUSETTS INSTITUTE OF TECHNOLOGY  
Department of Mechanical Engineering  
**2.004 Dynamics and Control II**  
Fall 2007

---

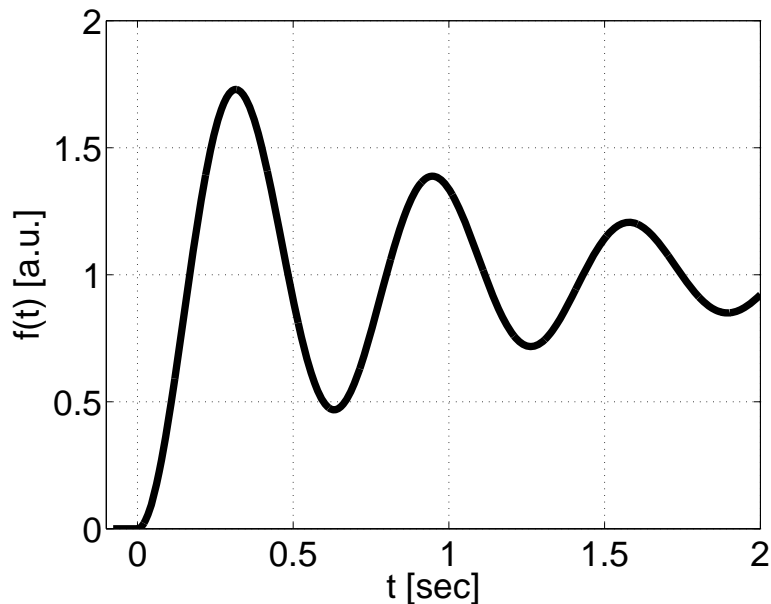
Problem Set #3

**Solution**

Posted: Friday, Sept. 28, '07

---

1. A second-order system has the step response shown below.<sup>1</sup> Determine its transfer function.



*Answer:* This is under-damped 2nd order system. Starting from the transfer function of the second order system

$$A \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2},$$

we have to decide the parameters of  $A$  (constant),  $\zeta$  (damping ratio) and  $\omega_n$  (natural frequency).

From the final value theorem,

$$\lim_{s \rightarrow 0} \frac{1}{s} \frac{A\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = A$$

---

<sup>1</sup>a.u. denotes arbitrary units; its use appropriate when we consider a function that does not correspond to any particular physical quantity.

and the steady state value is 1 (from the given figure). Therefore,  $A = 1$ .  
 The step response of the under-damped second order system is

$$[1 - ae^{-\sigma_d t} \cos(\omega_d t - \phi)] u(t),$$

where  $\sigma_d = \zeta\omega_n$  and  $\omega_d = \omega_n\sqrt{1 - \zeta^2}$ .

From the lecture note 7 (pp. 26), %OS =  $\exp\left(-\frac{\zeta\pi}{1-\zeta^2}\right) : 72\%$ .

Thus the damping ratio  $\zeta \approx 0.1$ .

To get the natural frequency, we choose two peak points at  $t_1 = 0.35$  sec and  $t_2 = 0.95$  sec. The cosine term will be 1 at the peaks, so that we can consider exponential decay term only.

$$\begin{aligned} f(t_1) &= 1 - ae^{-\sigma_d t_1} = 1.72 \\ f(t_2) &= 1 - ae^{-\sigma_d t_2} = 1.4 \end{aligned}$$

Dividing the two equations, we obtain

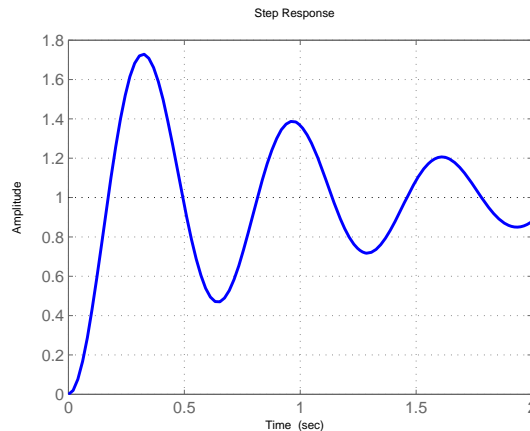
$$\frac{ae^{-\sigma_d t_1}}{ae^{-\sigma_d t_2}} = \frac{1 - 1.72}{1 - 1.4}.$$

From that  $\sigma_d = \left\{ \ln\left(\frac{0.72}{0.4}\right) \right\} / \{t_2 - t_1\} = 0.9796$ . Therefore  $\omega_n \approx 9.8$ . (The reason why I picked two points instead of one point is to cancel the constant  $a$ ).

The transfer function is

$$\frac{64}{s^2 + 1.96s + 96},$$

and its step response by MATLAB is



Note that the estimated parameters might be slightly different than the original because our reading of the plot can never be completely accurate.

2. Consider again the system of a DC motor with a parallel current source connected via a gear pair to an inertia that we saw in Problem 5 of PS02. Substituting numerical values  $i_s = 1.0u(t)$  A,  $R = 5 \Omega$ ,  $K_m = 0.5 \text{ N} \cdot \text{m}/\text{A}$ ,  $K_v = 0.5 \text{ V} \cdot \text{sec}$ ,  $J_m = 0.1 \text{ kg} \cdot \text{m}^2$ ,  $(N_2/N_1) = 10$ ,  $J = 6 \text{ kg} \cdot \text{m}^2$ ,  $K = 1 \text{ N} \cdot \text{m}/\text{rad}$ , derive and plot the step response for the following two cases:

a)  $b = 9.4 \text{ N} \cdot \text{m} \cdot \text{sec}/\text{rad}$ ;

b)  $b = 0.76 \text{ N} \cdot \text{m} \cdot \text{sec}/\text{rad}$ .

*Answer:*

The transfer function is

$$\frac{\Theta(s)}{I_s(s)} = \frac{(N_2/N_1)K_m}{\left(J + \frac{N_2^2}{N_1^2}J_m\right)s^2 + \left(b + \frac{N_2^2}{N_1^2}\frac{K_v K_m}{R}\right)s + K}$$

Rearranging it, we can re-write it as

$$\frac{\Theta(s)}{I_s(s)} = \frac{(N_2/N_1)K_m}{K} \frac{K/(J + (N_2/N_1)^2 J_m)}{s^2 + \frac{bR + (N_2^2/N_1^2)K_v K_m}{R(J + (N_2^2/N_1^2)J_m)}s + \frac{K}{J + (N_2^2/N_1^2)J_m}}$$

The general form of the transfer function is

$$\frac{\Theta(s)}{I_s(s)} = A \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2},$$

where the natural frequency  $\omega_n = \sqrt{\frac{K}{J + (N_2^2/N_1^2)J_m}}$  and the damping ratio  $\zeta = \frac{bR + (N_2^2/N_1^2)K_v K_m}{R(J + (N_2^2/N_1^2)J_m)} \frac{1}{2\omega_n}$ .

a)  $b = 9.4 \text{ N} \cdot \text{m} \cdot \text{sec}/\text{rad}$ ;

$\omega_n = 0.25$  and  $\zeta = 1.8 > 1$ . (Over-damped system) Transfer function is

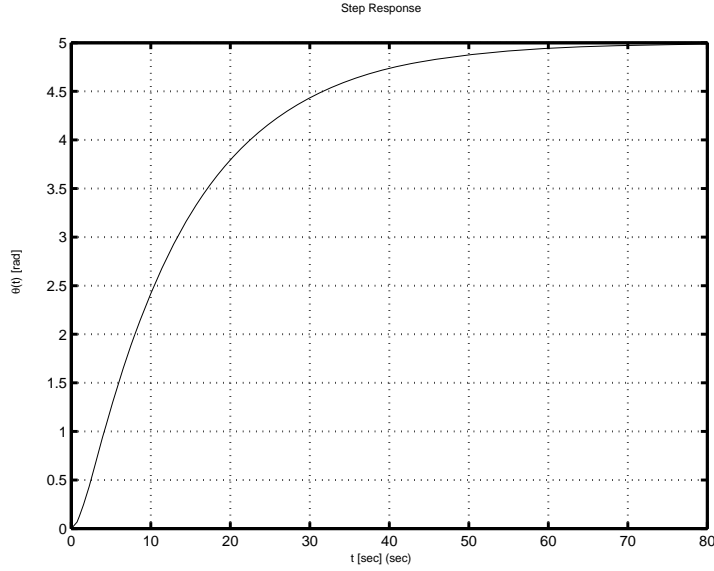
$$\frac{\Theta(s)}{I_s(s)} = \frac{5/16}{s^2 + (14.4/16)s + (1/4)^2}$$

whose poles are  $p_1 = -0.8242$  and  $p_2 = -0.0758$ . To obtain its step response, we do partial fraction expansion.

$$A \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{K_1}{s} + \frac{K_2}{s + 0.8242} + \frac{K_3}{s + 0.0758},$$

where  $K_1 = 5$ ,  $K_2 = 0.5067$ ,  $K_3 = -5.5067$ . The step response is

$$f(t) = (K_1 + K_2 e^{-0.8242t} + K_3 e^{-0.0758t})u(t).$$



b)  $b = 0.76 \text{ N} \cdot \text{m} \cdot \text{sec}/\text{rad}$ .

$\omega_n = 0.25$  and  $\zeta = 0.72 < 1$ . (Under-damped system) The transfer function is

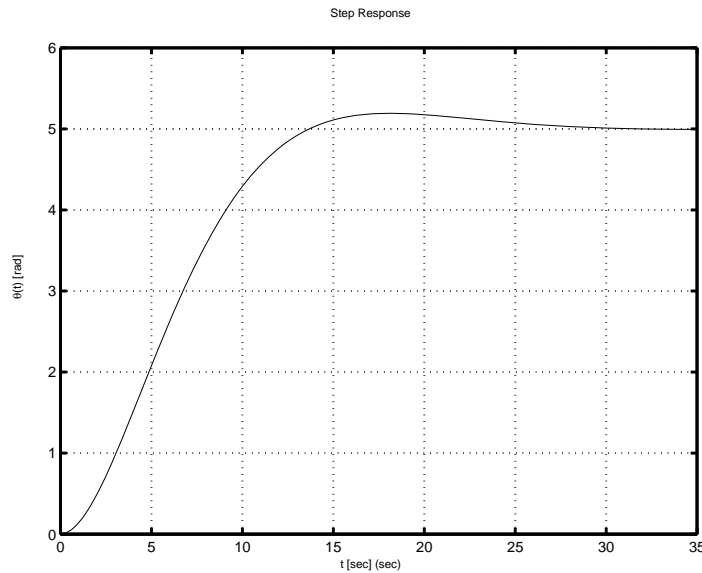
$$\frac{\Theta(s)}{I_s(s)} = \frac{5/16}{s^2 + (5.76/16)s + (1/4)^2}$$

whose poles are  $p_1 = -0.18 + j0.1735$  and  $p_2 = -0.18 - j0.1735$ . Doing partial fraction expansion, we obtain

$$A \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{K_1}{s} + \frac{K_2 s + K_3}{s^2 + 2\zeta\omega_n s + \omega_n^2},$$

where  $K_1 = 5$ ,  $K_2 = -5$ ,  $K_3 = -1.8$ . The step response is

$$f(t) = (K_1 + K_2 e^{-\sigma at} \cos(\omega_d t) + K_3 e^{-\sigma at} \sin(\omega_d t))u(t).$$

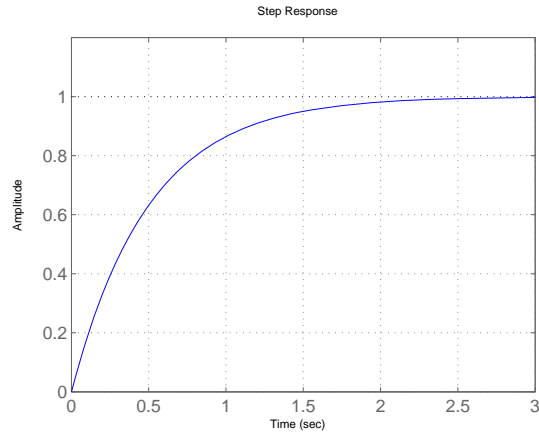
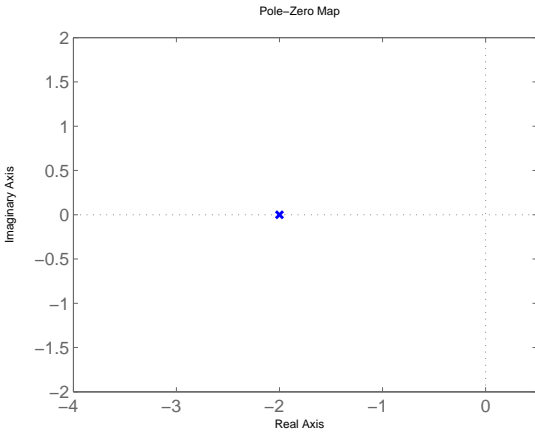


3. Problem 8 from Nise textbook, Chapter 4 (page 234).

*Answer: Plotting the step response was not required; we did it here for completeness.*

a)  $T(s) = \frac{2}{s+2}$

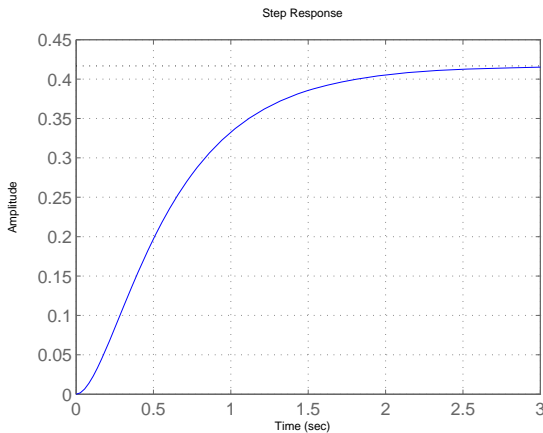
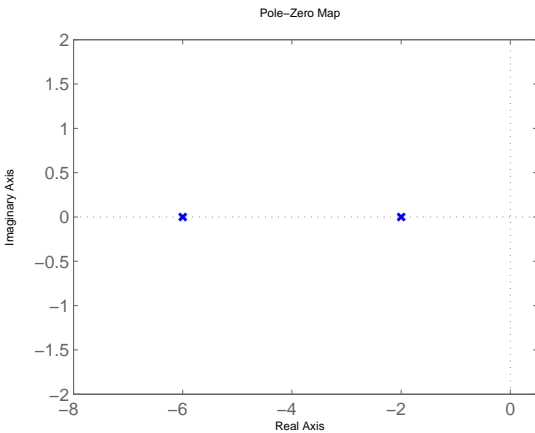
*Answer:* Pole:  $p=-2$ , Zero: none



Step response  $(1 - e^{-2t})u(t)$ . 1st order system.

b)  $T(s) = \frac{5}{(s+2)(s+6)}$

*Answer:* Poles:  $p_1 = -2, p_2 = -6$ , Zero: none

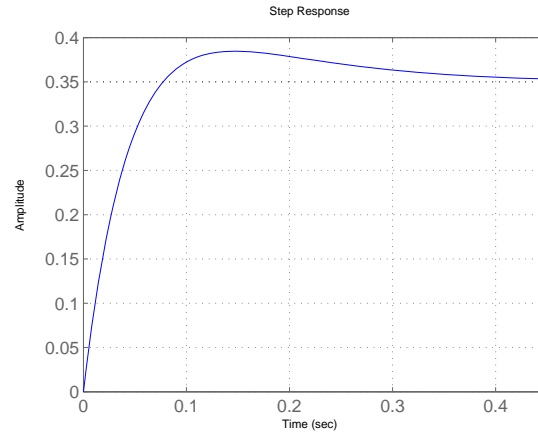
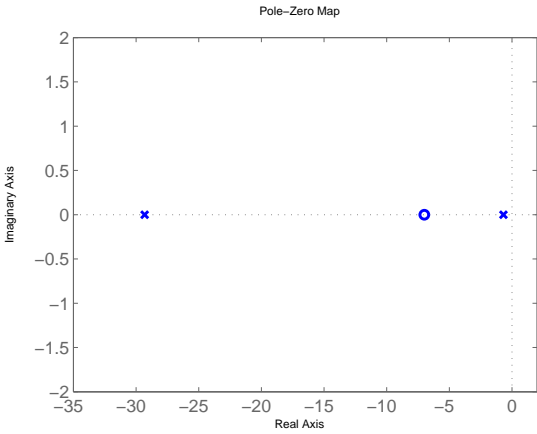


$\omega_n^2 = 12$  and  $\zeta = 4/\sqrt{12} = 1.15 > 1$ . 2nd order overdamped system.

Step response  $[1 + K_1e^{-p_1t} + K_2e^{-p_2t}]u(t)$ .

c)  $T(s) = \frac{10(s+7)}{(s+10)(s+20)}$

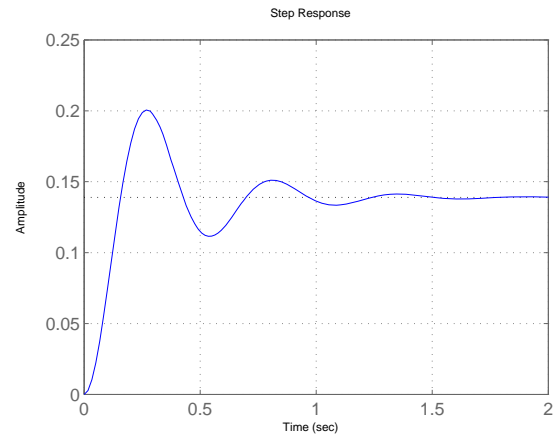
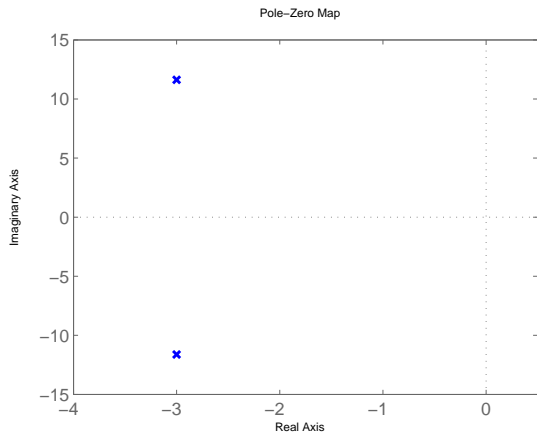
*Answer:* Poles:  $p_1 = -10, p_2 = -20$ , Zeros:  $z_1 = -7$



$\omega_n^2 = 200$  and  $\zeta = 30/2/\sqrt{200} = 1.06 > 1$ . 2nd order overdamped system.  
 Step response  $[1 + K_1 e^{-p_1 t} + K_2 e^{-p_2 t}] u(t)$ .

d)  $T(s) = \frac{20}{s^2 + 6s + 144}$

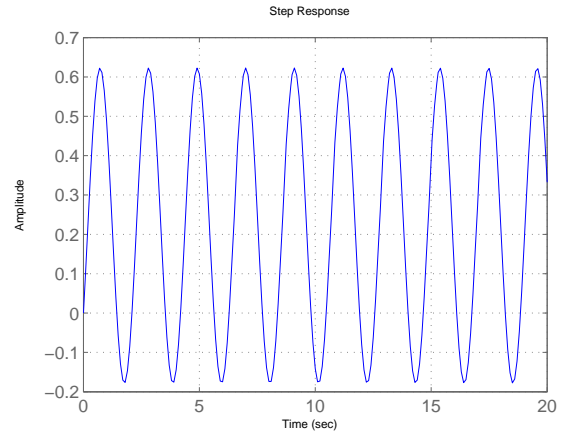
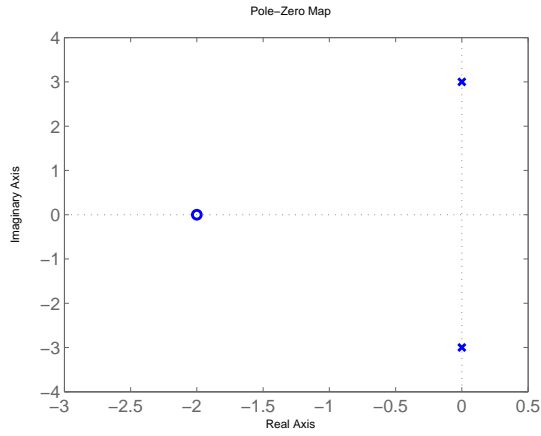
Answer: Poles:  $p_1 = -3 + j11.619$ ,  $p_2 = -3 - j11.619$ , Zeros: none



$\omega_n^2 = 144$  and  $\zeta = 6/2/\sqrt{144} = 0.25 < 1$ . 2nd order underdamped system.  
 Step response  $[1 - Ae^{-\sigma t} \cos(\omega_d t - \phi)] u(t)$ .

e)  $T(s) = \frac{s+2}{s^2+9}$

Answer: Poles:  $p_1 = 3j$ ,  $p_2 = -3j$ , Zero:  $z = -2$

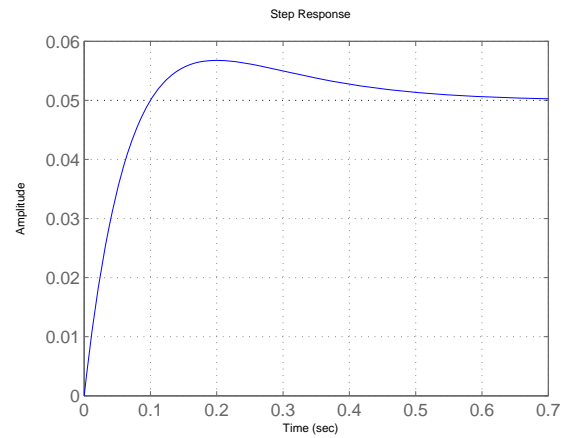
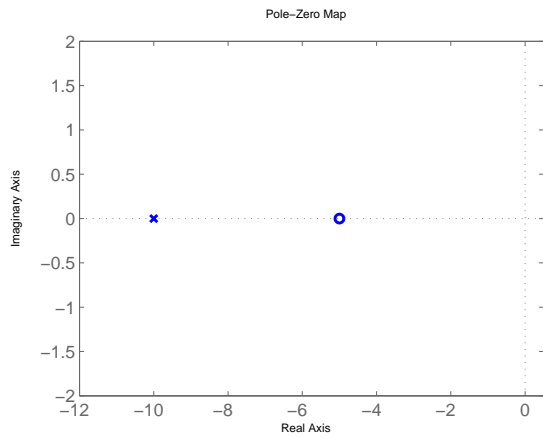


$\omega_n^2 = 9$  and  $\zeta = 0$ . 2nd order undamped system.

Step response  $[1 - K_1 \sin(3t) + K_2 \cos(3t)] u(t)$ .

**f)**  $T(s) = \frac{(s+5)}{(s+10)^2}$

Answer: Poles:  $p = -10$ (double), Zeors:  $z = -5$



$\omega_n^2 = 100$  and  $\zeta = 1$ . 2nd order critically damped system.

Step response  $[K_0 + K_1 e^{-10t} - K_2 t e^{-10t}] u(t)$ .