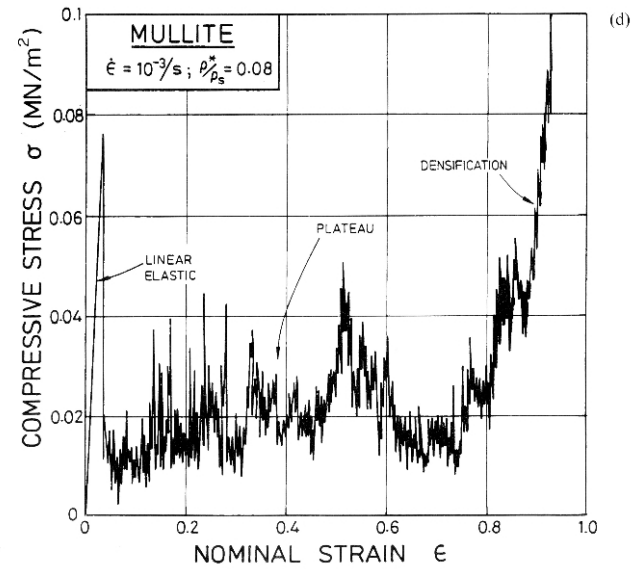
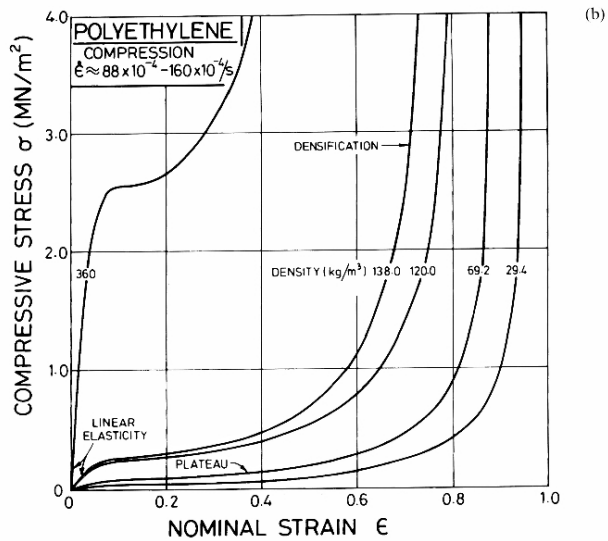
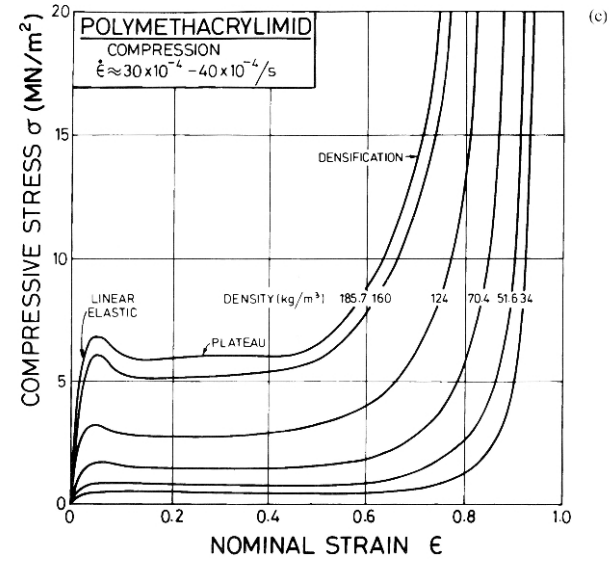
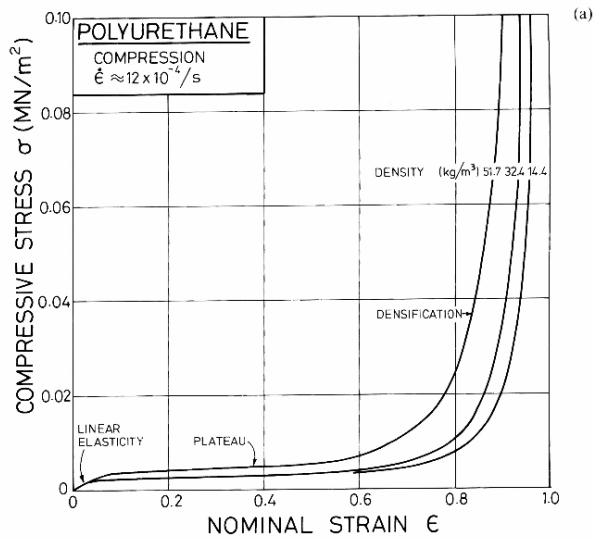


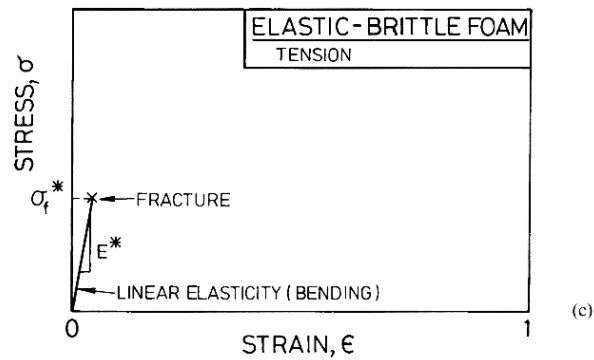
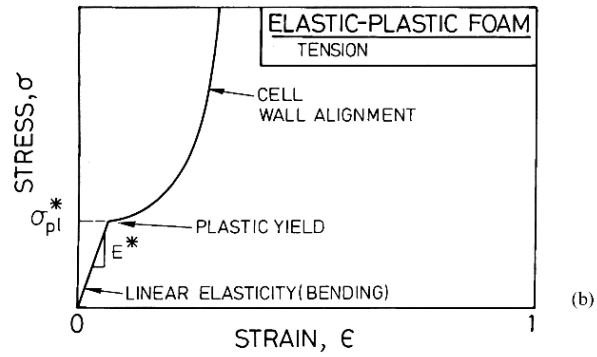
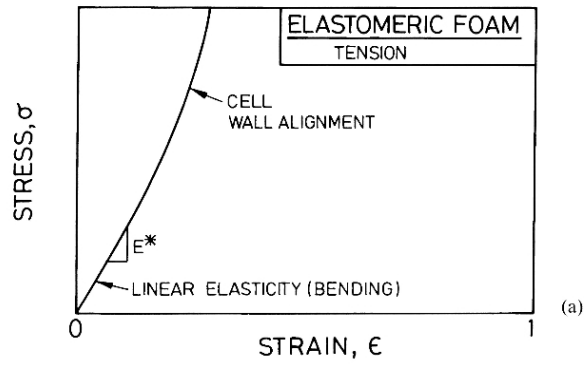
Open-cell foams

- stress-strain curves: deformation + failure mechanisms
 - compression - 3 regimes - linear elastic - bending
 - stress plateau - cell collapse by buckling
yielding
crushing
 - densification
 - tension - no buckling
 - yielding can occur
 - brittle fracture
-

Linear elastic behaviour

- initial linear elasticity - bending of cell edges (small t/l)
- as $t/l \uparrow$, axial deformation becomes more significant
- consider dimensional argument, which models mechanism of deformation + failure, but not cell geometry
- consider cubic cell, square cross-section members of area t^2 , length, l

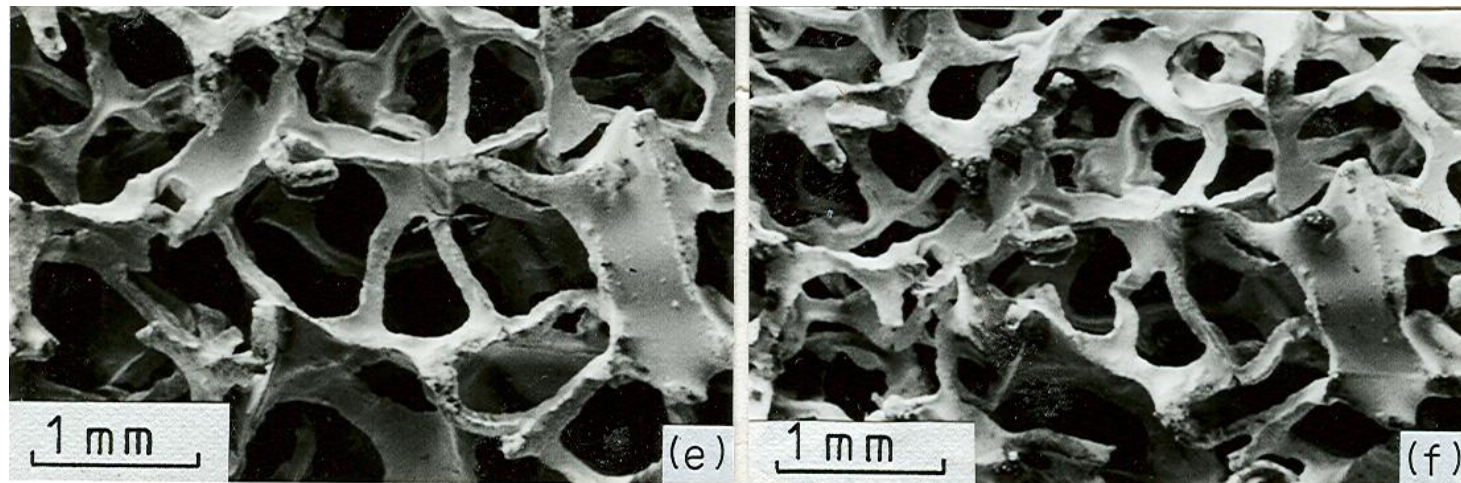




Foams: Bending, Buckling

Figure removed due to copyright restrictions. See Fig. 3: Gibson, L. J., and M. F. Ashby. "[The Mechanics of Three-Dimensional Cellular Materials](#)." *Proceedings of The Royal Society of London A* 382 (1982): 43-59.

Foams: Plastic Hinges



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

Foams: Cell Wall Fracture

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- regardless of specific cell geometry chosen:

$$\rho^*/\rho_s \propto (t/l)^2 \quad I \propto t^4$$

$$\sigma \propto F/l^2 \quad E \propto \delta/l \quad \delta \propto Fl^3/E_s I$$

$$E^* \propto \frac{\sigma}{E} \propto \frac{F}{l^2} \frac{l}{\delta} \propto \frac{F}{l} \frac{E_s t^4}{Fl^3} \propto E_s \left(\frac{t}{l}\right)^4 \propto E_s \left(\frac{\rho^*}{\rho_s}\right)^2$$

$$\boxed{E^* = C_1 E_s (\rho^*/\rho_s)^2}$$

C_1 includes all geometrical constants.

Data: $C_1 \approx 1$

- data suggest $C_1 = 1$
- analysis of open cell tetrakaidecahedral cells with Plateau borders gives $C_1 = 0.98$
- shear modulus $\boxed{G^* = C_2 E_s (\rho^*/\rho_s)^2}$ $C_2 \sim 3/8$ if foam isotropic

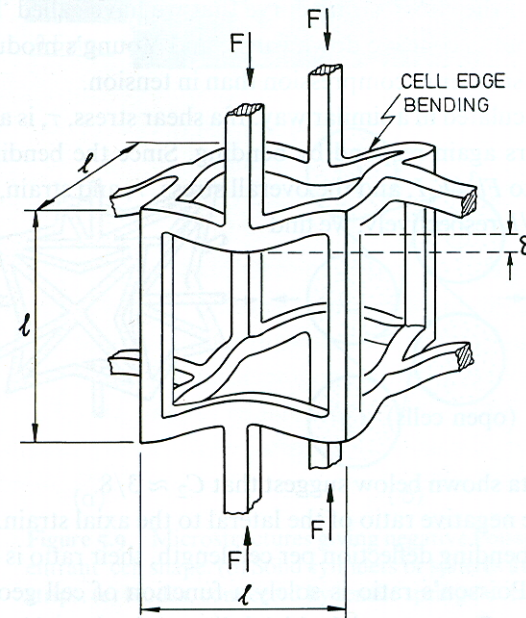
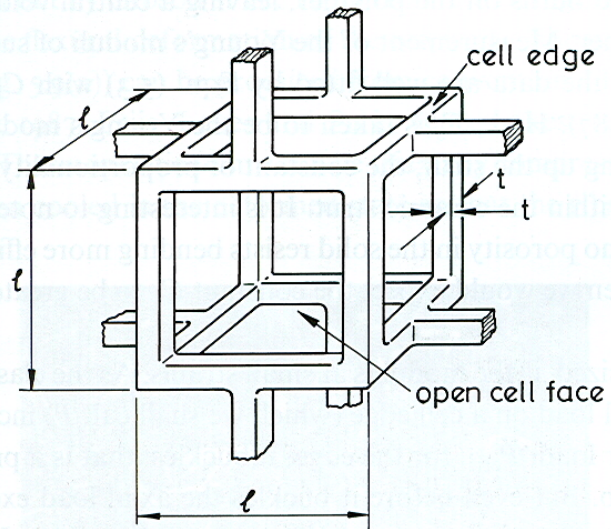
$$\text{isotropy: } G = \frac{E}{2(1+\nu)}$$

- Poisson's ratio $\nu^* = \frac{E}{2G} - 1 = \frac{C_1}{2C_2} - 1 = \text{constant, independent of } E_s, t/l$

$$\boxed{\nu^* = C_3}$$

(analogous to honeycombs in-plane)

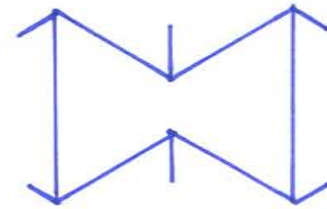
Foam: Edge Bending



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Poisson's ratio

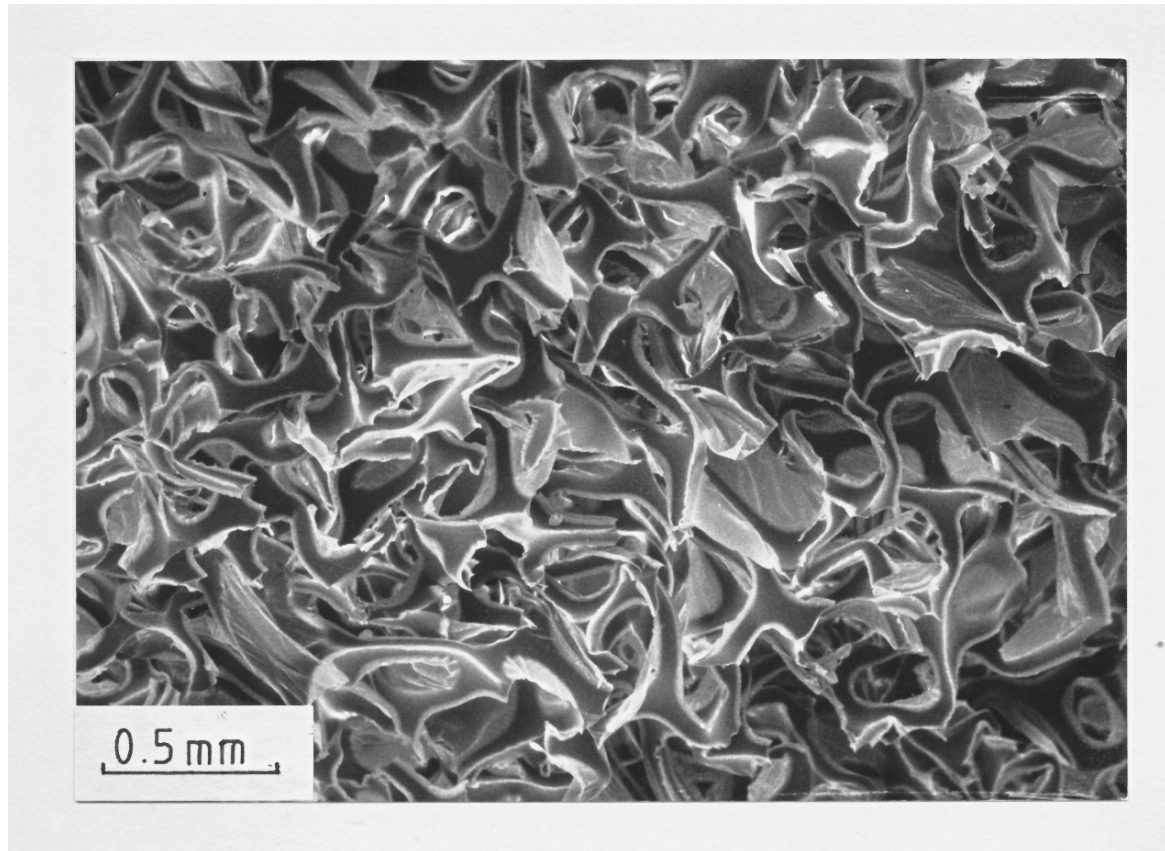
- can make negative Poisson's ratio foams
- invert cell angles (analogous to honeycomb)
- eg. thermoplastic foams - load hydrostatically + heat to $T > T_g$, then cool + release load so that edges of cell permanently point inward.



Closed-cell foams

- edge bending as for open-cell foams
- also: face stretching + gas compression
- polymer foams: surface tension draws material to edges during processing
 - define t_e , t_f in figure
- apply F to the cubic structure

Negative Poisson's Ratio



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

- external work done $\propto F\delta$
- internal work bending edges $\propto \frac{F_e}{\delta_e} \delta_e^2 \propto \frac{E_s I}{l^3} \delta^2$
- internal work stretching faces $\propto \sigma_f \epsilon_f V_f \propto E_s \epsilon_f^2 V_f \propto E_s \left(\frac{\delta}{l}\right)^2 t_f l^2$

$$\therefore F\delta = \alpha \frac{E_s t_e^4}{l^3} \delta^2 + \beta E_s \left(\frac{\delta}{l}\right)^2 t_f l^2$$

$$E^* \propto \frac{F}{l^2} \frac{l}{\delta} \Rightarrow F \propto E^* \delta l$$

$$\therefore E^* \delta l = \alpha \frac{E_s t_e^4}{l^3} \delta^2 + \beta E_s \left(\frac{\delta}{l}\right)^2 t_f l^2$$

$$E^* = \alpha E_s \left(\frac{t_e}{l}\right)^4 + \beta E_s \left(\frac{t_f}{l}\right)$$

Note: open cells, uniform t :

$$\rho^*/\rho_s \propto (t/l)^2$$

if ϕ = volume fraction of solid in cell edges :

$$t_e/l = C \phi^{1/2} (\rho^*/\rho_s)^{1/2}$$

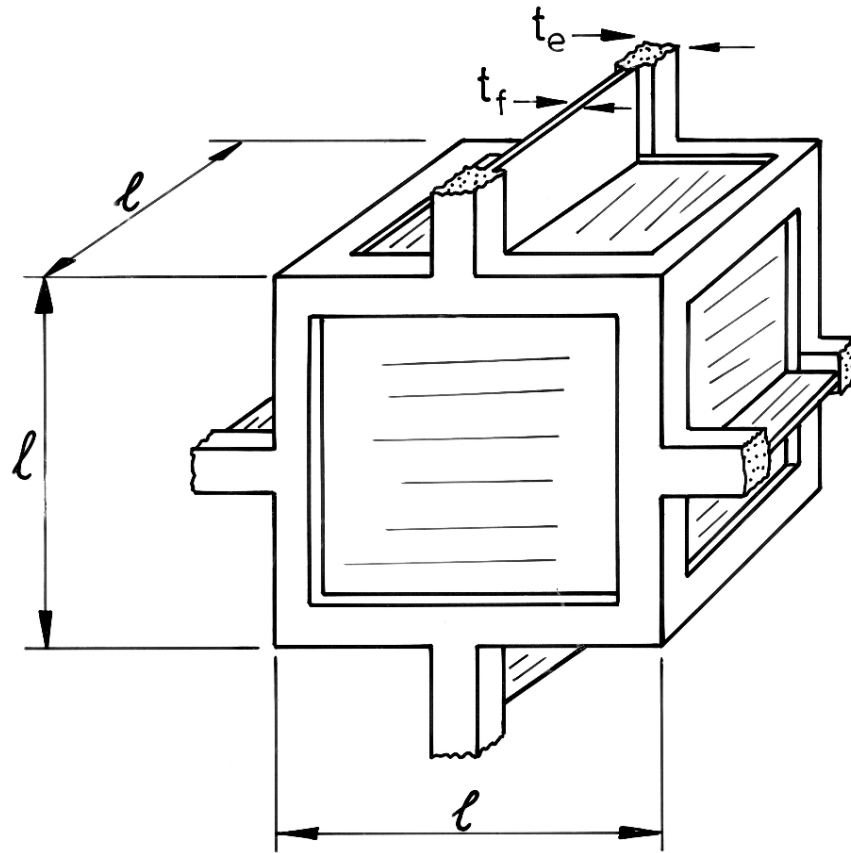
closed cells, uniform t

$$\rho^*/\rho_s \propto (t/l)$$

$$t_f/l = C' (1-\phi) (\rho^*/\rho_s)$$

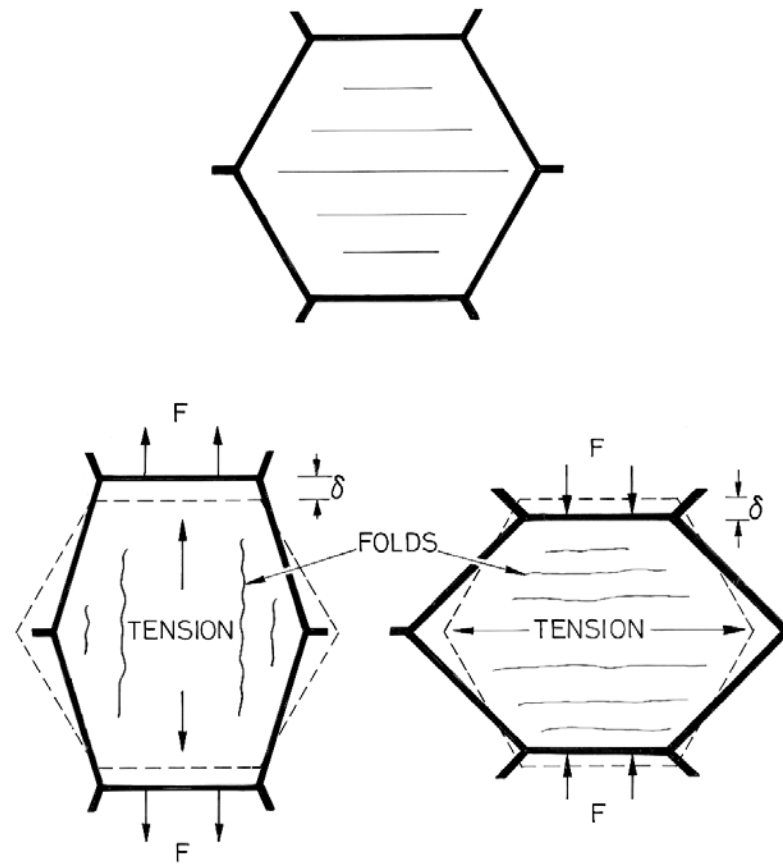
$$\boxed{\frac{E^*}{E_s} = C_1 \phi^2 \left(\frac{\rho^*}{\rho_s}\right)^2 + C_1' (1-\phi) \frac{\rho^*}{\rho_s}}$$

Closed-Cell Foam



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Cell Membrane Stretching



Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

Closed cell foams - gas within cell may also contribute to E^*

- cubic element of foam of volume V_0
- under uniaxial stress, axial strain in direction of stress is ϵ
- deformed volume V is:

$$\frac{V}{V_0} = 1 - \epsilon(1 - 2\nu^*)$$

taking compressive strain as positive
neglecting ϵ^2, ϵ^3 terms

$$\frac{V_g}{V_g^0} = \frac{1 - \epsilon(1 - 2\nu^*) - \rho^*/\rho_s}{1 - \rho^*/\rho_s}$$

V_g = volume gas
 V_g^0 = " " initially

- Boyle's law: $pV_g = p_0V_g^0$

p = pressure after strain ϵ
 p_0 = pressure initially

- pressure that must be overcome is $p' = p - p_0$

$$p' = \frac{p_0 \epsilon (1 - 2\nu^*)}{1 - \epsilon(1 - 2\nu^*) - \rho^*/\rho_s}$$

- contribution of gas compression to the modulus, E_g^* :

$$E_g^* = \frac{dp'}{d\epsilon} = \frac{p_0 (1 - 2\nu^*)}{1 - \rho^*/\rho_s}$$

(4a)



$$V_0 = l_0^3$$

$$V = l_1 l_2 l_3$$

$$\epsilon_1 = \frac{l_1 - l_0}{l_0} \Rightarrow l_1 = l_0 + \epsilon_1 l_0 = l_0 (1 + \epsilon_1)$$

$$\epsilon_2 = \frac{l_2 - l_0}{l_0} \Rightarrow l_2 = l_0 + \epsilon_2 l_0 \quad \nu = -\frac{\epsilon_2}{\epsilon_1}$$

$$= l_0 - \nu \epsilon_1 l_0$$

$$\epsilon_2 = -\nu \epsilon_1$$

$$= l_0 (1 - \nu \epsilon_1)$$

$$\epsilon_3 = l_0 (1 - \nu \epsilon_1)$$

$$V = l_1 l_2 l_3 = l_0 (1 + \epsilon_1) l_0 (1 - \nu \epsilon_1) l_0 (1 - \nu \epsilon_1) = l_0^3 (1 + \epsilon_1) (1 - \nu \epsilon_1)^2$$

$$\frac{V}{V_0} = \frac{l_0^3 (1 + \epsilon_1) (1 - \nu \epsilon_1)^2}{l_0^3} = (1 + \epsilon_1) (1 - 2\nu \epsilon_1 + \nu^2 \epsilon_1^2)$$

$$= (1 - 2\nu \epsilon_1 + \nu^2 \epsilon_1^2) + \epsilon_1 - 2\nu \epsilon_1^2 + \nu^2 \epsilon_1^3$$

$$= 1 + \epsilon_1 + 2\nu \epsilon_1^2 \quad (\text{taking comp. as +})$$

$$= 1 - \epsilon_1 (1 - 2\nu)$$

(56)

$$P' = P - P_0$$

$$P = \frac{P_0 V_g^0}{V_g}$$

$$P' = P - P_0 = \frac{P_0 V_g^0}{V_g} - P_0 = P_0 \left(\frac{V_g^0}{V_g} - 1 \right) \quad \left(\frac{V_g^0 - V_g}{V_g} \right)$$

$$= P_0 \left[\frac{1 - p/p_s}{1 - \epsilon(1 - 2v^*) - p^*/p_s} - 1 \right]$$

$$= P_0 \left[\frac{1 - p/p_s}{1 - \epsilon(1 - 2v^*) - p^*/p_s} - \frac{1 - \epsilon(1 - 2v^*) - p^*/p_s}{1 - \epsilon(1 - 2v^*) - p^*/p_s} \right]$$

$$= P_0 \left[\frac{\epsilon(1 - 2v^*)}{1 - \epsilon(1 - 2v^*) - p^*/p_s} \right] \quad \checkmark$$

Closed cell foam

$$\frac{E^*}{E_s} = \underbrace{\phi^2 \left(\frac{\rho^*}{\rho_s}\right)^2}_{\text{edge bending}} + \underbrace{(1-\phi) \left(\frac{\rho^*}{\rho_s}\right)}_{\text{face stretching}} + \underbrace{\frac{p_0 (1-2\nu^*)}{E_s (1-\rho^*/\rho_s)}}_{\text{gas compression}}$$

- note: if $p_0 = p_{atm} = 0.1 \text{ MPa}$, gas compression term negligible, except for closed-cell elastomeric foams
- gas comp. can be significant if $p_0 \gg p_{atm}$; also modifies shape of stress plateau in elastomeric closed cell foams

Shear modulus: edge bending, face stretching; shear $\Delta V = 0$ gas contrib. = 0

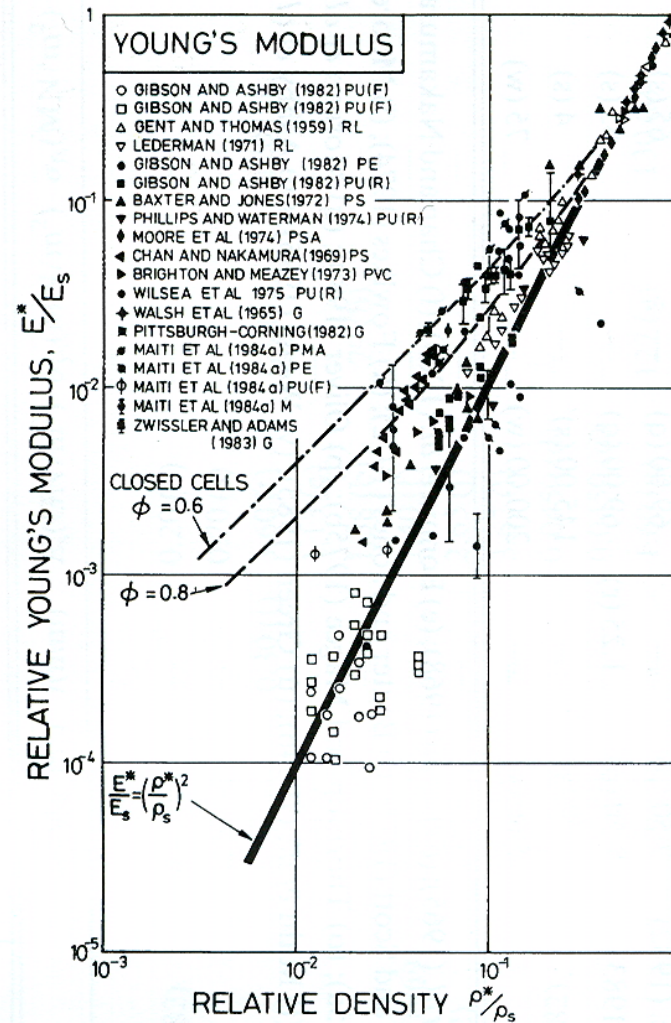
$$\frac{G^*}{E_s} = \frac{3}{8} \left[\phi^2 \left(\frac{\rho^*}{\rho_s}\right)^2 + (1-\phi) \left(\frac{\rho^*}{\rho_s}\right) \right] \quad (\text{isotropic foam})$$

Poisson's ratio = f (cell geometry only) $\nu^* \approx 1/3$

Comparison with data

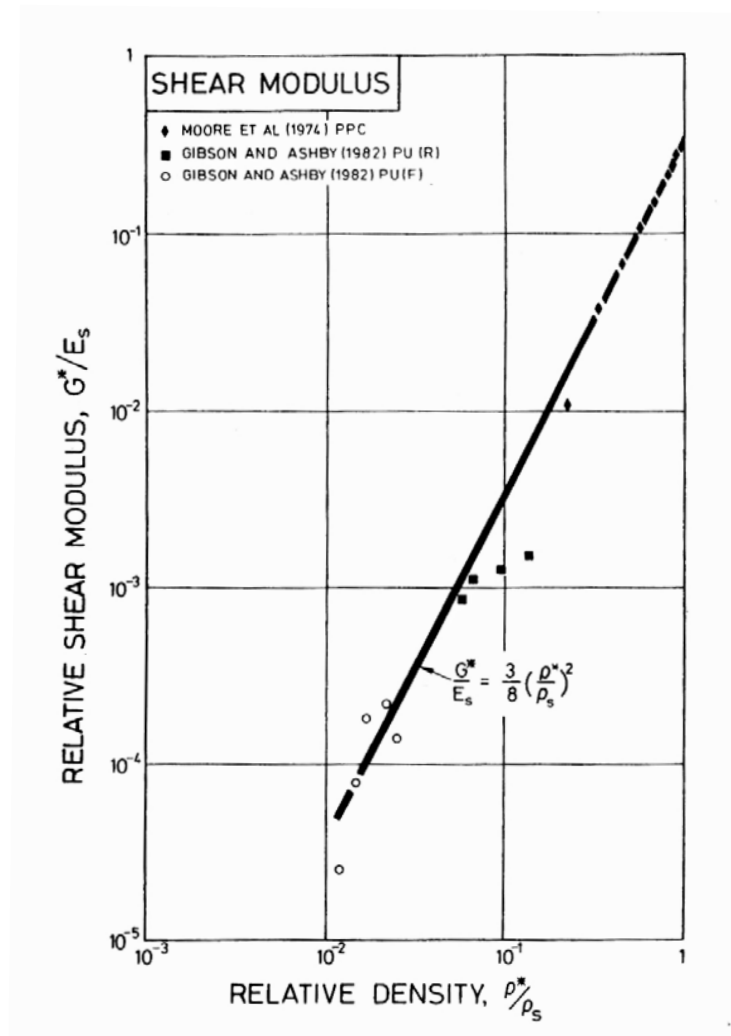
- data for polymers, glasses, elastomers
- E_s, ρ_s Table 5.1 in book
- open cells - open symbols
- closed cells - filled symbols

Young's Modulus



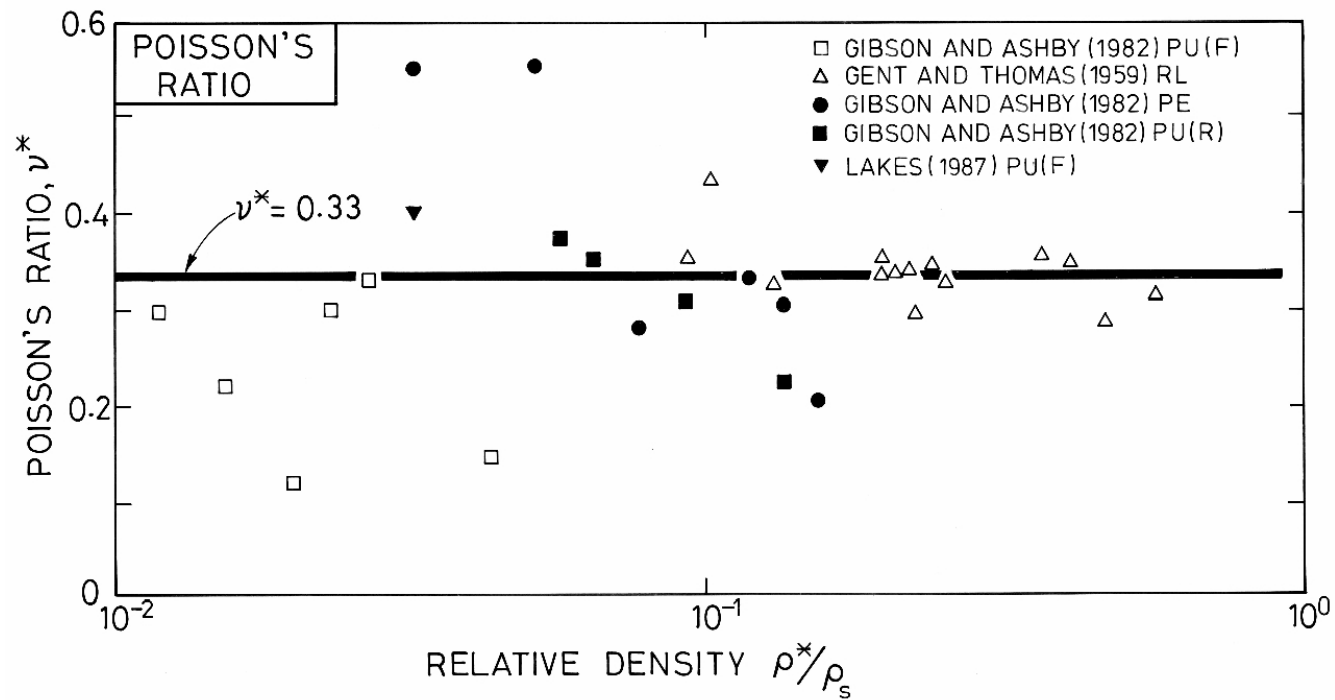
Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

Shear Modulus



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Poisson's Ratio



Non-linear elasticity

Open cells:

$$P_{cr} = \frac{n^2 \pi^2 E_s I}{l^2}$$

$$\sigma_{el}^* \propto \frac{P_{cr}}{l^2} \propto E_s \left(\frac{t}{l}\right)^4$$

$$\sigma_{el}^* = C_4 E_s (\rho^*/\rho_0)^2$$

Data: $C_4 \approx 0.05$, corresponds to strain when buckling initiates, since $E^* = E_s (\rho^*/\rho_0)^2$

Closed cells

- t_f often small compared to t_e (surface tension in processing) - contrib. small
- if $p_0 \gg p_{atm}$, cell walls pretensioned; buckling stress has to overcome this

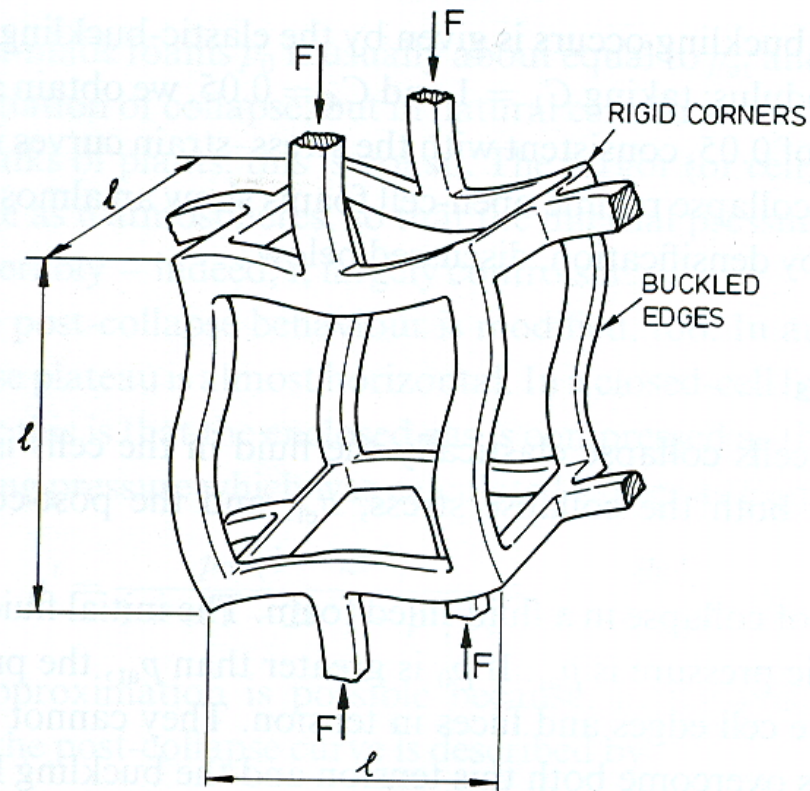
$$\sigma_{el}^* = 0.05 E_s (\rho^*/\rho_0)^2 + p_0 - p_{atm}$$

- post-collapse behaviour - stress plateau rises due to gas compression (if faces don't rupture) $v^* = 0$ in post-collapse regime

$$p' = \frac{p_0 \epsilon (1 - 2v^*)}{1 - \epsilon(1 - 2v^*) - \rho^*/\rho_0} = \frac{p_0 \epsilon}{1 - \epsilon - \rho^*/\rho_0}$$

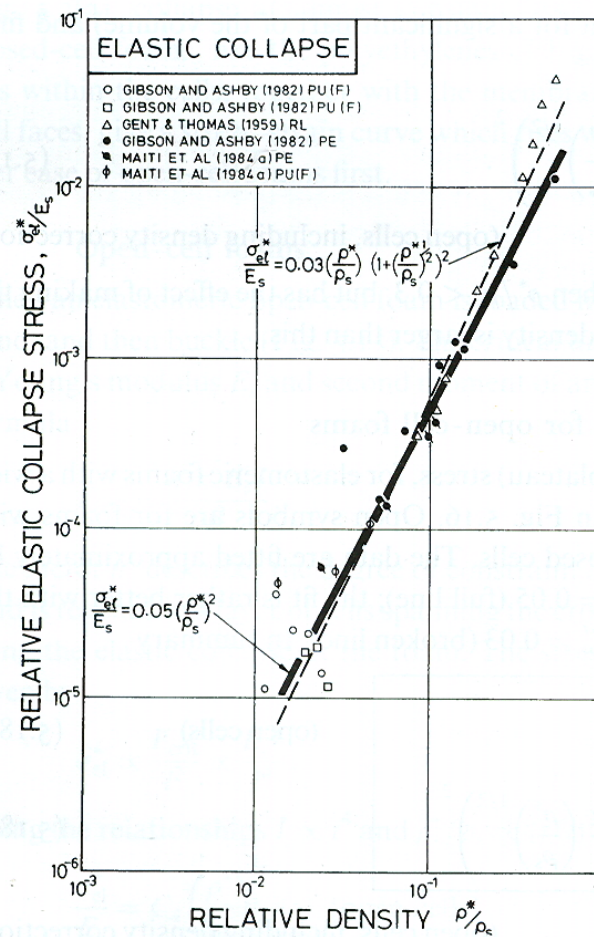
$$\sigma_{post-collapse}^* = 0.05 E_s \left(\frac{\rho^*}{\rho_0}\right)^2 + \frac{p_0 \epsilon}{1 - \epsilon - \rho^*/\rho_0}$$

Elastic Collapse Stress



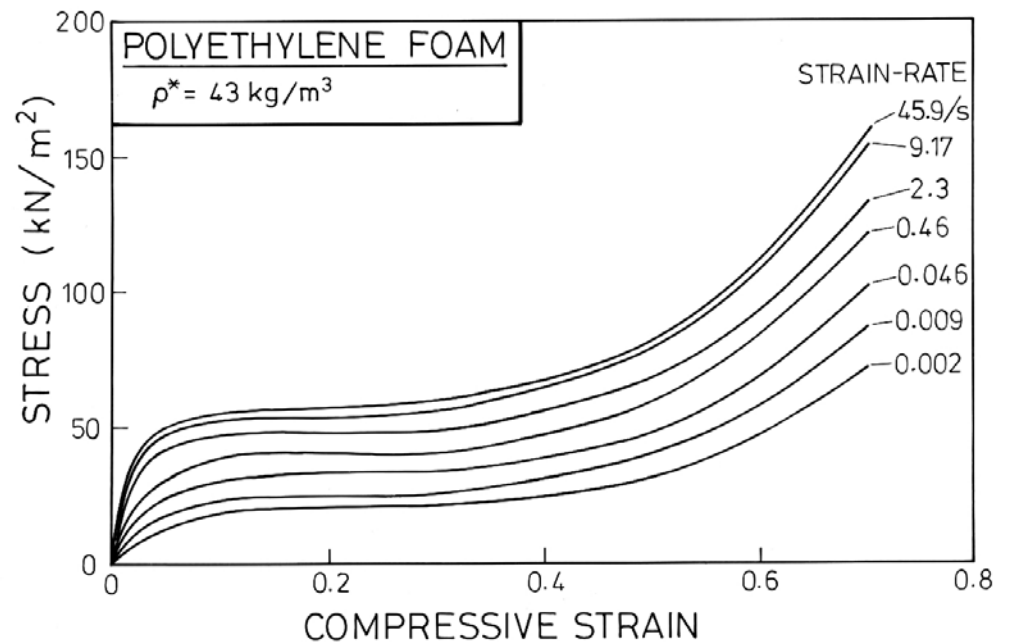
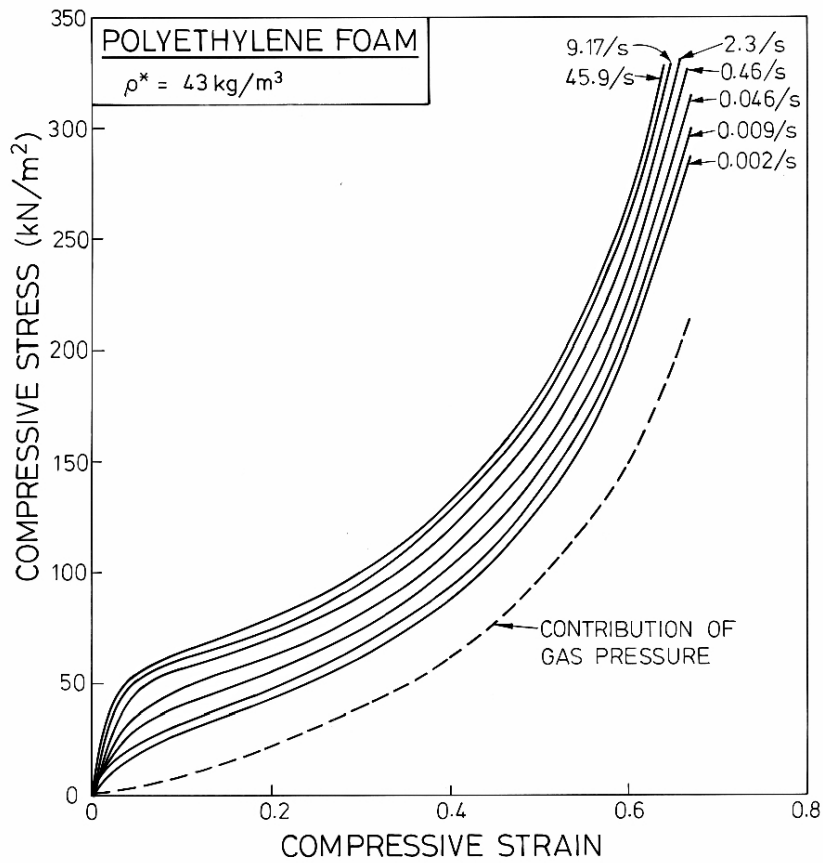
Gibson, L. J., and M. F. Ashby. *Cellular Solids: Structure and Properties*. 2nd ed. Cambridge University Press, © 1997. Figure courtesy of Lorna Gibson and Cambridge University Press.

Elastic Collapse Stress

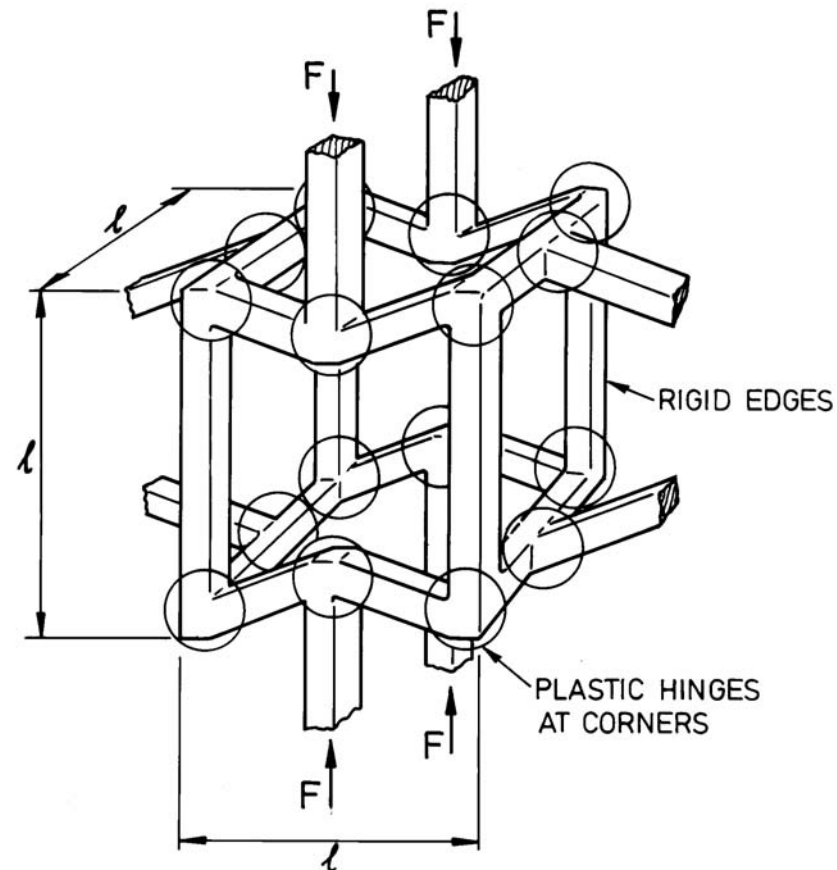


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Post-collapse stress strain curve

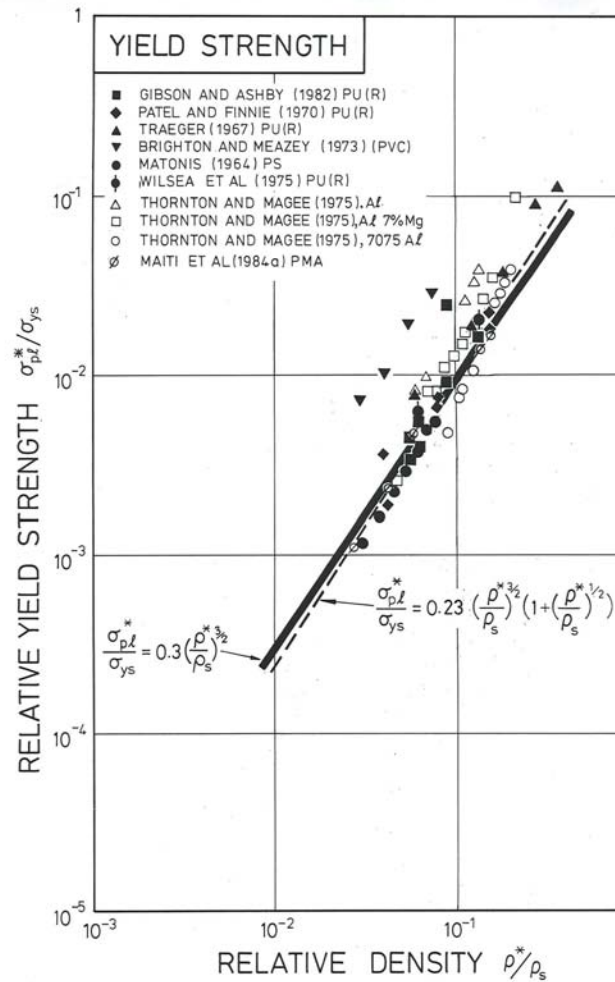


Plastic Collapse Stress



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Plastic Collapse Stress



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Brittle crushing strength

Open cells

- failure when $M = M_f$

$$\sigma_{cr}^* = C_6 \sigma_{fs} (\rho^*/\rho_s)^{3/2}$$

$$M \propto \sigma_{cr}^* l^3$$

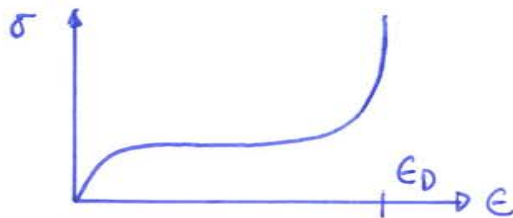
$$M_f \propto \sigma_{fs} t^3$$

$$C_6 \approx 0.2$$

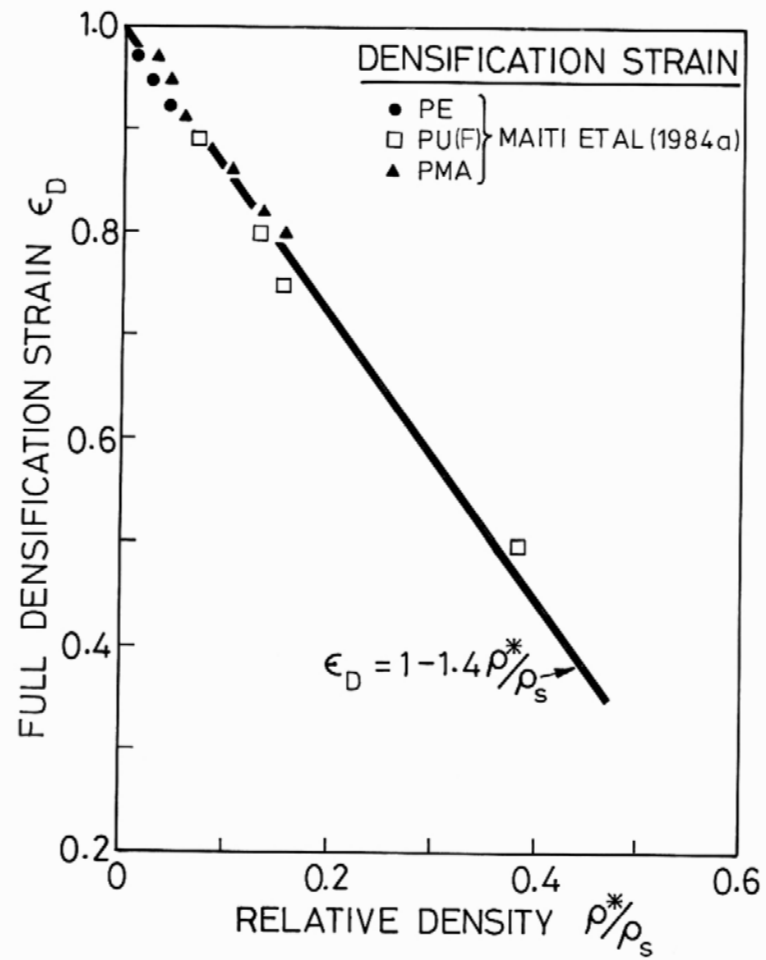
Densification strain, ϵ_D

- at large comp. strain, cell walls begin to touch, σ - ϵ rises steeply
- $E^* \rightarrow E_s$; σ - ϵ curve looks vertical, at limiting strain
- might expect $\epsilon_D = 1 - \rho^*/\rho_s$
- walls jam together at slightly smaller strain than this:

$$\epsilon_D = 1 - 1.4 \rho^*/\rho_s$$



Densification Strain



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Fracture toughness

Open cells: crack length $2a$, local stress σ_l , remote stress σ^∞

$$\sigma_l = \frac{C \sigma^\infty \sqrt{\pi a}}{\sqrt{2\pi r}} \quad \text{a distance } r \text{ from crack tip}$$

- next unbroken cell wall a distance $r \approx l/2$ ahead of crack tip subject to a force (integrating stress over next cell)

$$F \propto \sigma_l l^2 \propto \sigma^\infty \sqrt{\frac{a}{l}} l^2$$

- edges fail when applied moment, $M = \text{fracture moment}, M_f$

$$M_f \propto \sigma_{fs} t^3$$

$$M \propto F l^2 \propto \sigma^\infty \left(\frac{a}{l}\right)^{1/2} l^3$$

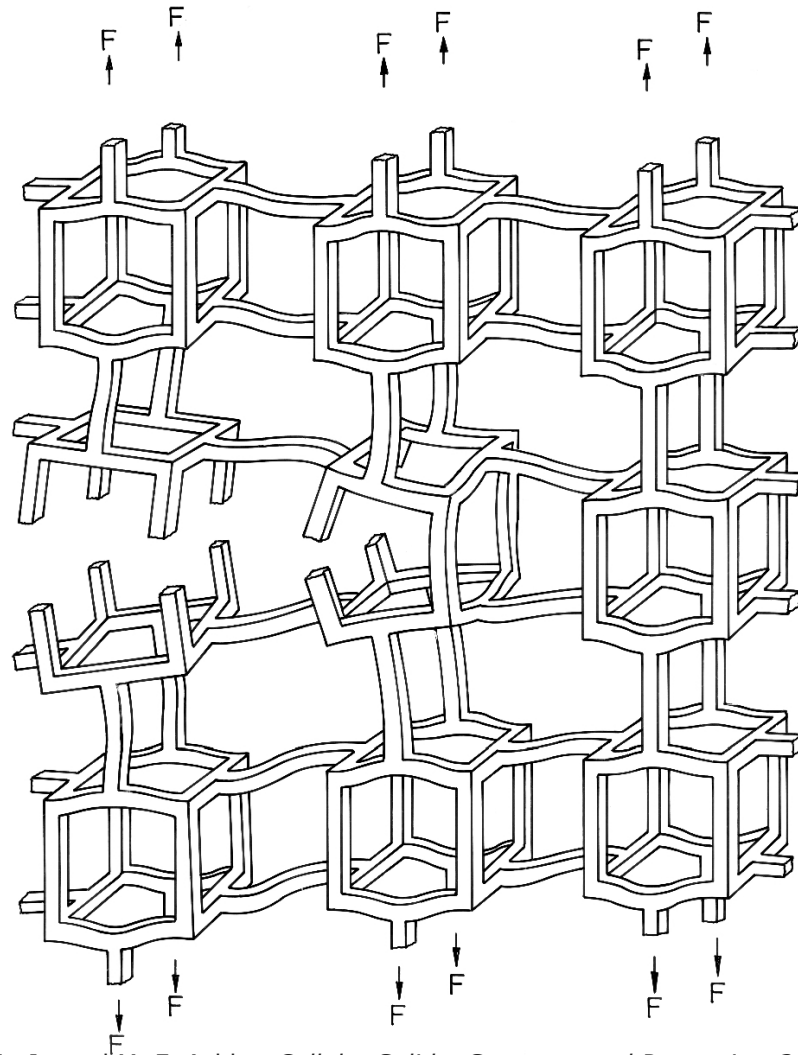
$$M = M_f \Rightarrow \sigma^\infty \left(\frac{a}{l}\right)^{1/2} l^3 \propto \sigma_{fs} t^3$$

$$\sigma^\infty \propto \sigma_{fs} \left(\frac{l}{a}\right)^{1/2} \left(\frac{t}{l}\right)^3$$

$$\boxed{K_{Ic}^* = \sigma^\infty \sqrt{\pi a} = C_g \sigma_{fs} \sqrt{\pi l} (\rho^*/\rho_s)^{3/2}}$$

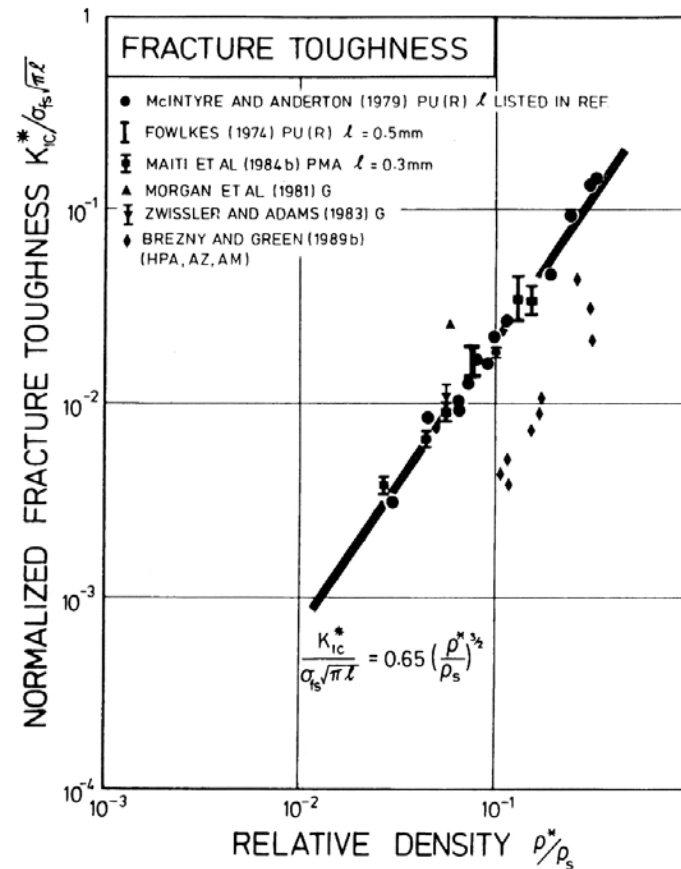
Data: $C_g \sim 0.65$

Fracture Toughness



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Fracture Toughness



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