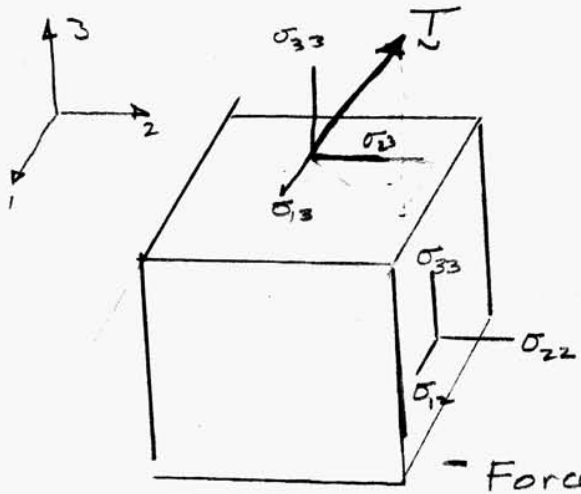


# Stress



Traction  $\vec{T}$  on material may depend on area

Body forces  $\vec{g}$  depend on volume neglect in this discussion

- Forces inside body which are a reaction to traction are stresses
- Homogeneous if forces acting on a surface of fixed shape and orientation do not depend on position in body.

Then traction on each face of unit-cube can be characterized.

For body to be in static equilibrium, Traction on (+2 face) = (-2 face) and opposite in dir otherwise net imbalance causes acceleration governed by Newton's Law.

Define stress on each face as traction / area

$\Rightarrow \vec{T} = \underline{\underline{\sigma}} \underline{\underline{A}}$  i.e. stress tensor relates area normal  $\rightarrow$  traction

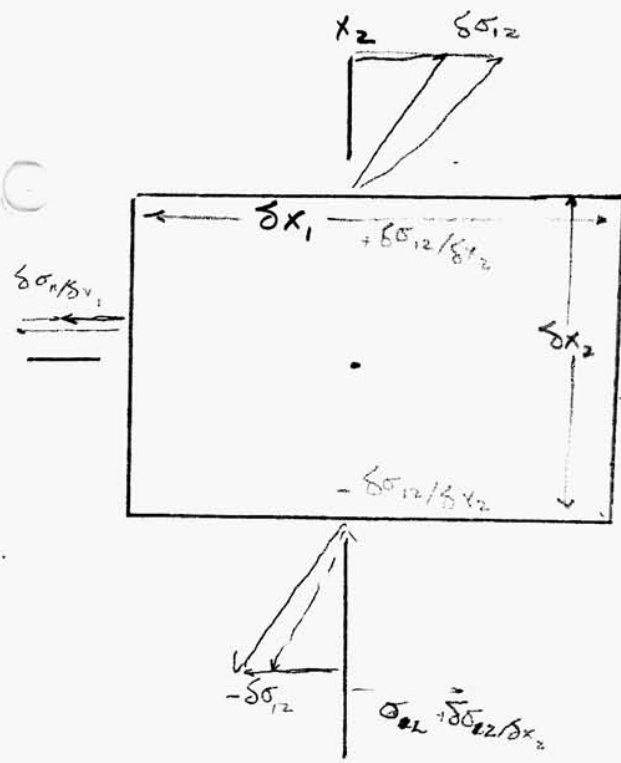
$$\vec{T}_i = \sigma_{ij} A_j$$

$\sigma_{ij} \triangleq T_i$  on  $j$  face

- normal components,  $\sigma_{ii}$  + if traction
- shear components  $\sigma_{ij}, i \neq j$  + if in positive dir on " face
- + if in negative dir on " face



Egn. Equilibrium



On  $x_1$  face in  $x_1$  direct

$$\text{left } \sigma_{11} \delta x_2 \delta x_3 + \frac{\partial \sigma_{11}}{\partial x_1} \frac{\delta x_1}{2} \delta x_2 \delta x_3$$

$$\text{right } - \left[ \sigma_{11} \delta x_2 \delta x_3 - \frac{\partial \sigma_{11}}{\partial x_1} \frac{\delta x_1}{2} \delta x_2 \delta x_3 \right]$$

result.  $\frac{\partial \sigma_{11}}{\partial x_1} (\delta x_1 \delta x_2 \delta x_3)$

on  $x_2$  in  $x_1$  direction

$$\text{top } \sigma_{12} (\delta x_1 \delta x_3) + \frac{\partial \sigma_{12}}{\partial x_2} \frac{\delta x_2}{2} (\delta x_1 \delta x_3)$$

$$\text{bottom } - \left[ \sigma_{12} (\delta x_1 \delta x_3) + \frac{\partial \sigma_{12}}{\partial x_2} \left( -\frac{\delta x_2}{2} \right) (\delta x_1 \delta x_3) \right]$$

$$\frac{\partial \sigma_{12}}{\partial x_2} (\delta x_1 \delta x_2 \delta x_3)$$

By Newton's law, sum of forces in  $x_i$  direction equals mass/vol  $\times$  acceleration in that direction

$$\frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \sigma_{12}}{\partial x_2} + \frac{\partial \sigma_{13}}{\partial x_3} (+ \rho g_1) = \rho \ddot{x}_1$$

In general,

$$\frac{\partial \sigma_{ij}}{\partial x_j} + \rho g_i = \rho \ddot{x}_i$$

If

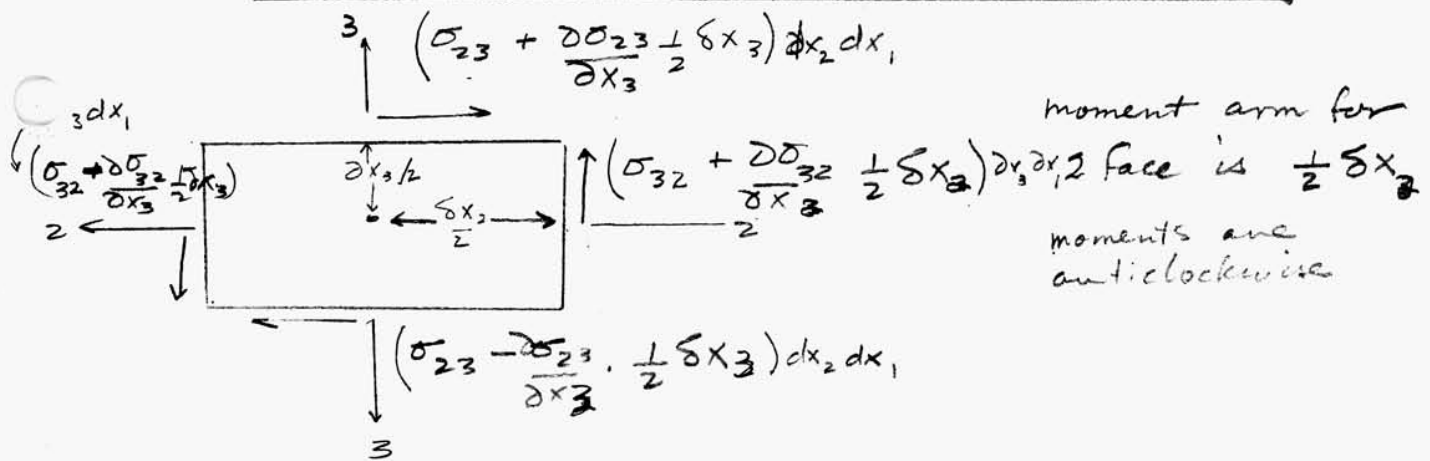
$$x_i = 0, \text{ i.e.}$$

in static eq.

$$\frac{\partial \sigma_{ij}}{\partial x_j} + \rho g_i = 0$$

# 3 of 6

## Shear moments and symmetry



moment arm for  
2 face is  $\frac{1}{2} \delta x_3$   
moments are  
anticlockwise

moment arm for 3 face is  $\frac{1}{2} \delta x_3$   
moments are clockwise

$$2\sigma_{32} \frac{\delta x_3}{2} (dx_2 dx_1) - 2\sigma_{23} \frac{dx_2}{2} dx_3 dx_1 + G_1 dx_1 dx_2 dx_3 = I_1 \ddot{\theta}_1$$

but assume no body torques  $G_1 = 0$

and note  $I_1$  order of mag  $\rho \delta x^5 \rightarrow 0$  as  $dx \downarrow$

then  $(\sigma_{32} - \sigma_{23}) dx_1 dx_2 dx_3 = 0$

$$\Rightarrow \sigma_{32} = \sigma_{23}$$

or  $\sigma_{ij} = \sigma_{ji}$

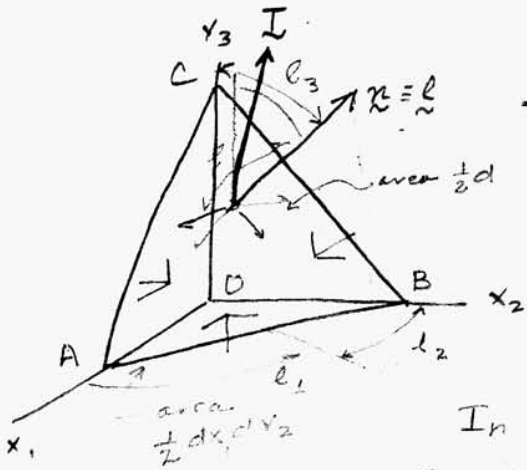
$\Rightarrow$  Stress tensor is symmetric.

Proof that stress is a tensor

Cauchy tetrahedron

Total force on face with area ABC

$$\vec{T} = \vec{P} \cdot (\text{area } ABC)$$



In  $x_1$  direction

$$P_1(ABC) = \sigma_{11}(BOC) + \sigma_{12}(AOC) + \sigma_{13}(AOB)$$

$$P_1 = \sigma_{11} l_1 + \sigma_{12} l_2 + \sigma_{13} l_3$$

where  $l_1 = \frac{\text{area } OBC}{\text{area } ABC} = \cos \angle x_1, \vec{n}$

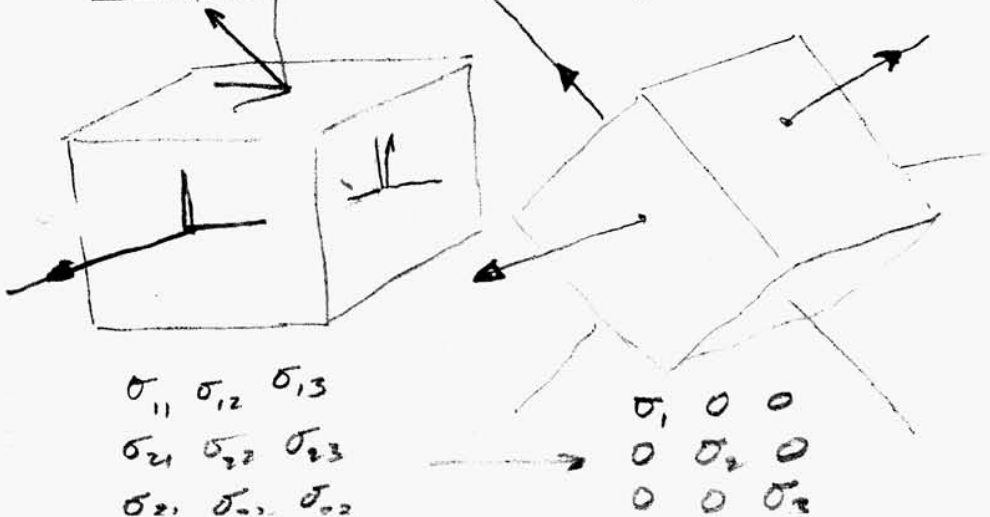
sim.  $P_2 = \sigma_{21} l_1 + \sigma_{22} l_2 + \sigma_{23} l_3$

$P_3 = \sigma_{31} l_1 + \sigma_{32} l_2 + \sigma_{33} l_3$

or  $P_i = \sigma_{ij} l_j$   
 i.e. tensor transformation rule

⇒ Stress is a 2nd rank, symmetric tensor

⇒ Principal stresses, Quadratic



# Special Forms of Stress

5 of 6

i.) uniaxial stress

$$\begin{bmatrix} \sigma & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

ii.) biaxial stress

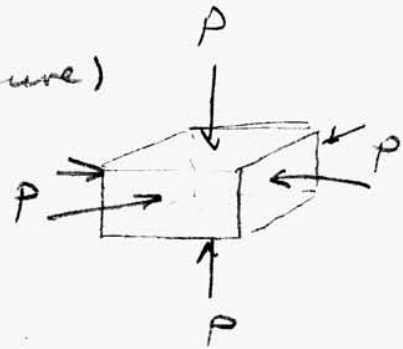
$$\begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

iii.) triaxial stress

$$\begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}$$

iv.) hydrostatic stress (pressure)

$$\begin{bmatrix} -P & 0 & 0 \\ 0 & -P & 0 \\ 0 & 0 & -P \end{bmatrix} = -P\delta_{ij}$$



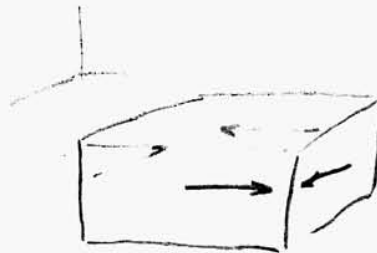
v.) Pure shear

$$\begin{bmatrix} -\sigma & 0 & 0 \\ 0 & \sigma & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

see above

vi.) Simple shear

$$\begin{bmatrix} 0 & \sigma & 0 \\ \sigma & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$



Rod stretch

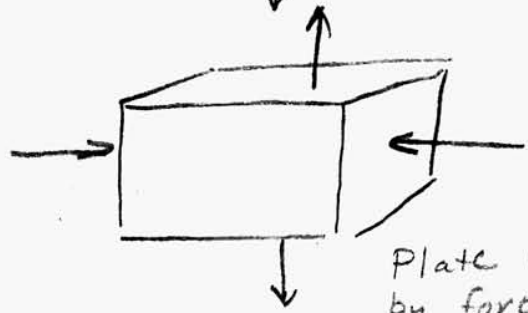
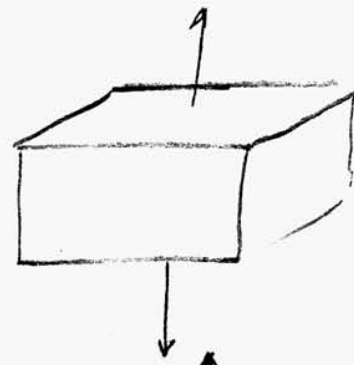


Plate loaded by forces  $\epsilon_{comp}$  at end  
 $\Delta x_1, \Delta x_2 \ll x_3$



## Summary: Stress Tensor

1.) Traction on a plane with direction cosine  $\underline{l}$

$$\underline{T} = \underline{\underline{\sigma}} \underline{l} \quad T_i = \sigma_{ij} l_j$$

2.) Eqs of motion

$$\frac{\partial \sigma_{ij}}{\partial x_i} + \rho g_j = m \ddot{x}_j$$

3.) Stress is symmetric

$$\sigma_{ij} = \sigma_{ji}$$

4.) Principal stress directions  
values

$$\begin{array}{ccc} \sigma_I & 0 & 0 \\ 0 & \sigma_{II} & 0 \\ 0 & 0 & \sigma_{III} \end{array}$$

5.) Stress quadric construction

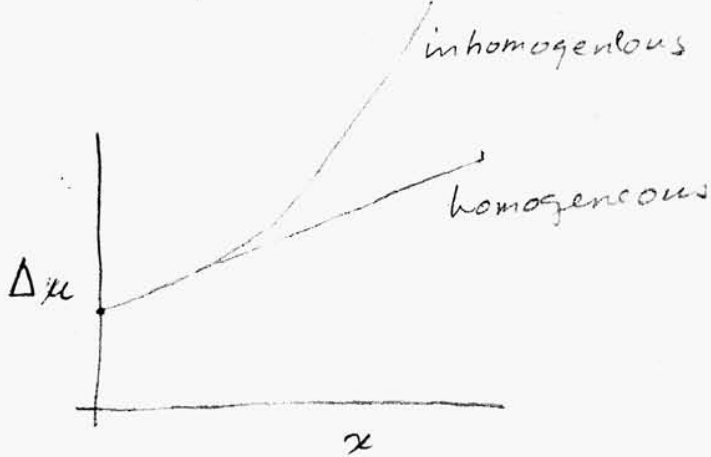
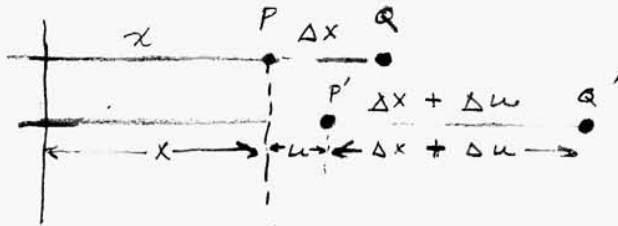
# Stress, Strain, Elasticity:

1 of 8

Nye, Chap 5 & 6, pps 82-105.

## 1. One dimensional strain.

Relative displacements important



Strain at a point P is

$$\epsilon = \frac{\text{increase in length}}{\text{original length}} = \frac{P'Q' - PQ}{PQ} = \frac{\Delta u}{\Delta x}$$

(slope of curve above)

Goal: For a given deformation of body, Define a tensor that describes the change of direction and length of any vector in body.

Want to map vector in body (undef) into vector in body (def.)