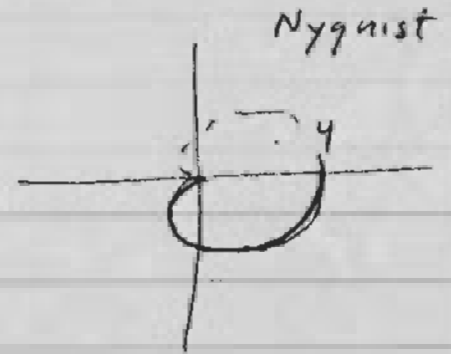


ohi



$$G(s) = \frac{k}{(s+1)^2}$$

$$k=4$$

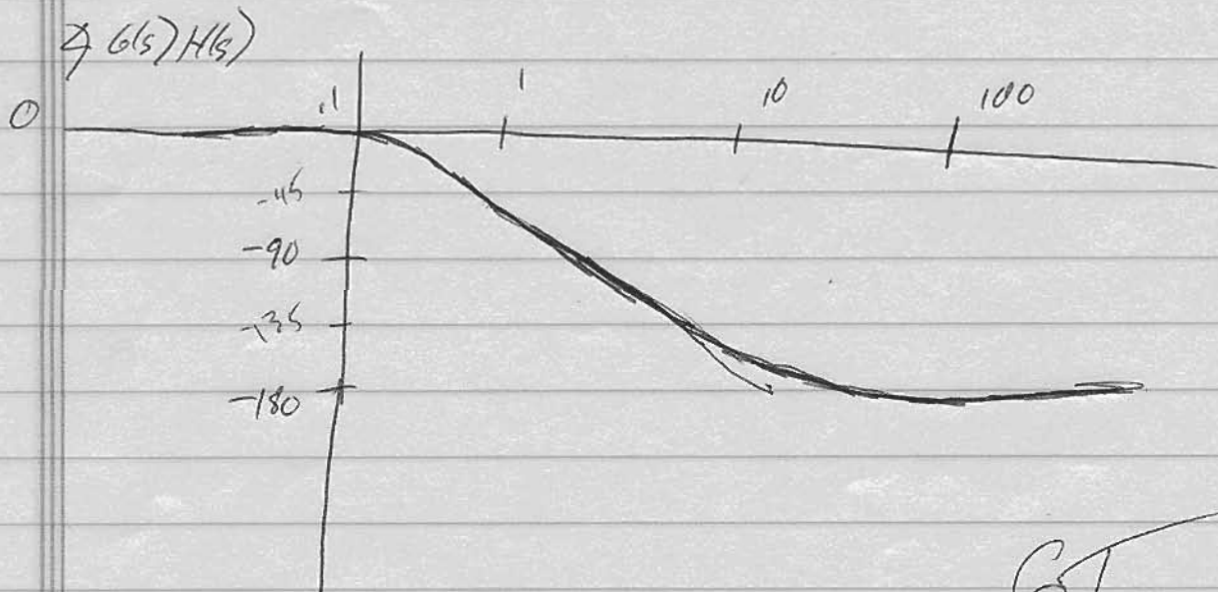
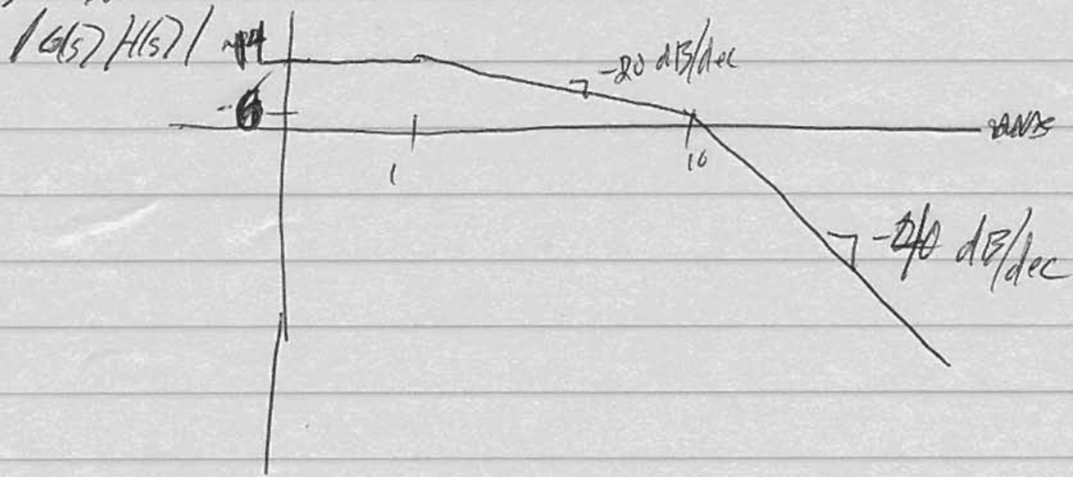


ω	$M_{\omega} = \frac{4}{\omega^2 + 1}$	Phase $-2 \tan^{-1} \left(\frac{\omega}{1} \right)$	
		deg	rad
0.5	3.2	-53.13	-0.9273
1	2	-90	-1.5708
2	0.8	-126.87	-2.2143
4	0.2353	-151.93	-2.6576

Glenn

2. $G(s)H(s) = \frac{50}{s^2 + 11s + 10}$

- a.) corner freq ~~ω_n~~ $\omega = 1, 10$
b.) at low freq slope = 0 (0 dB/dec)
high freq slope = -2 (-40 dB/dec)
c.) Bode Plot



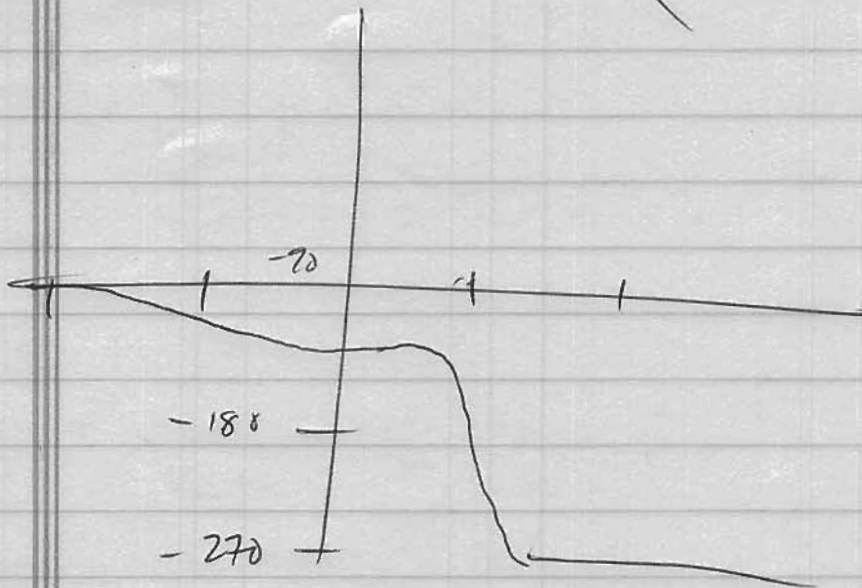
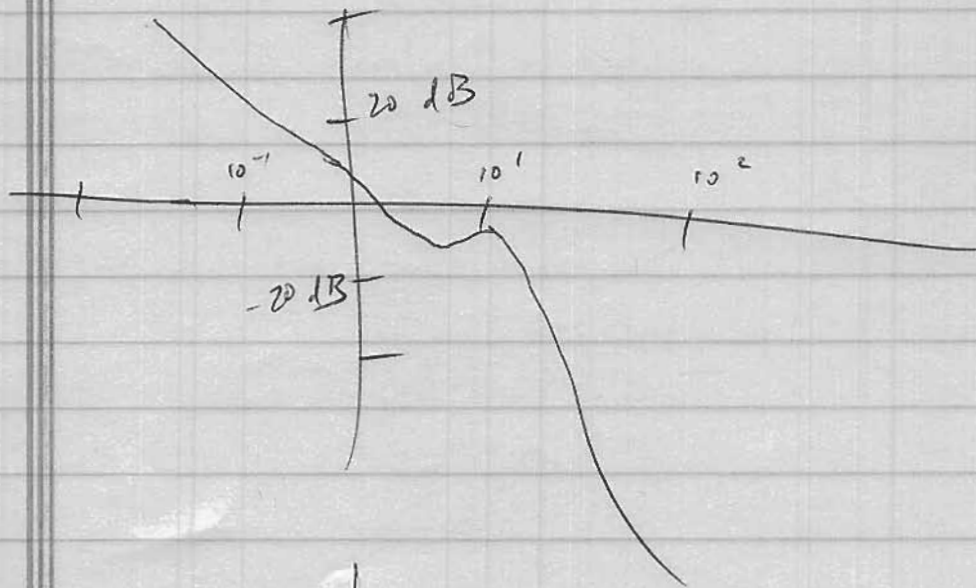
GT

phil

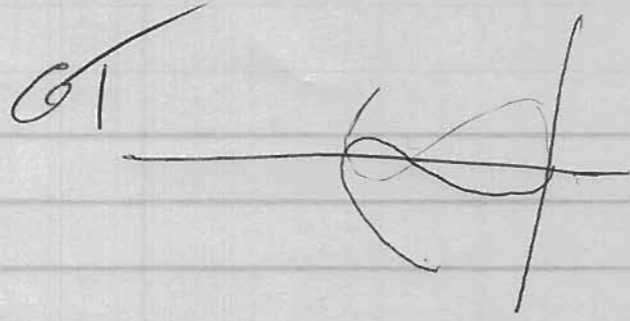
GM 9.33 dB @ 7.7 rad/s

PM 51.2° @ 1.57 rad/s

(3)



#4



$$4.) C(s) = \frac{0.30(s+0.05)(s^2+1600)}{(s^2+0.05s+16)(s+70)}$$

$$a.) K(s) = 2 \quad H(s) = 0.5$$

$$g_m = \infty$$

$$\phi_m = 79.3^\circ \quad 13.2 \text{ rad/sec}$$

$$b.) K(s) = K_1 + K_2/s \quad K_2/K_1 = 0.5$$
$$= \frac{K_1 s + K_2}{s}$$
$$= \frac{K_2 (K_1/K_2 s + 1)}{s}$$

$$\omega = 4$$

$$\omega = 0.16$$

$$10^{-1/2} = \frac{0.30 \cdot (\omega^2 + 0.05^2)^{1/2} (1600 - \omega^2) ((2 \cdot \omega)^2 + 1)^{1/2}}{\omega ((16 - \omega^2)^2 + (0.05\omega)^2)^{1/2} (70^2 + \omega^2)^{1/2}}$$

$$\omega = 0.16$$

$$K_2 = .669$$

$$K_1 = 2K_2$$

$$\omega = 4$$

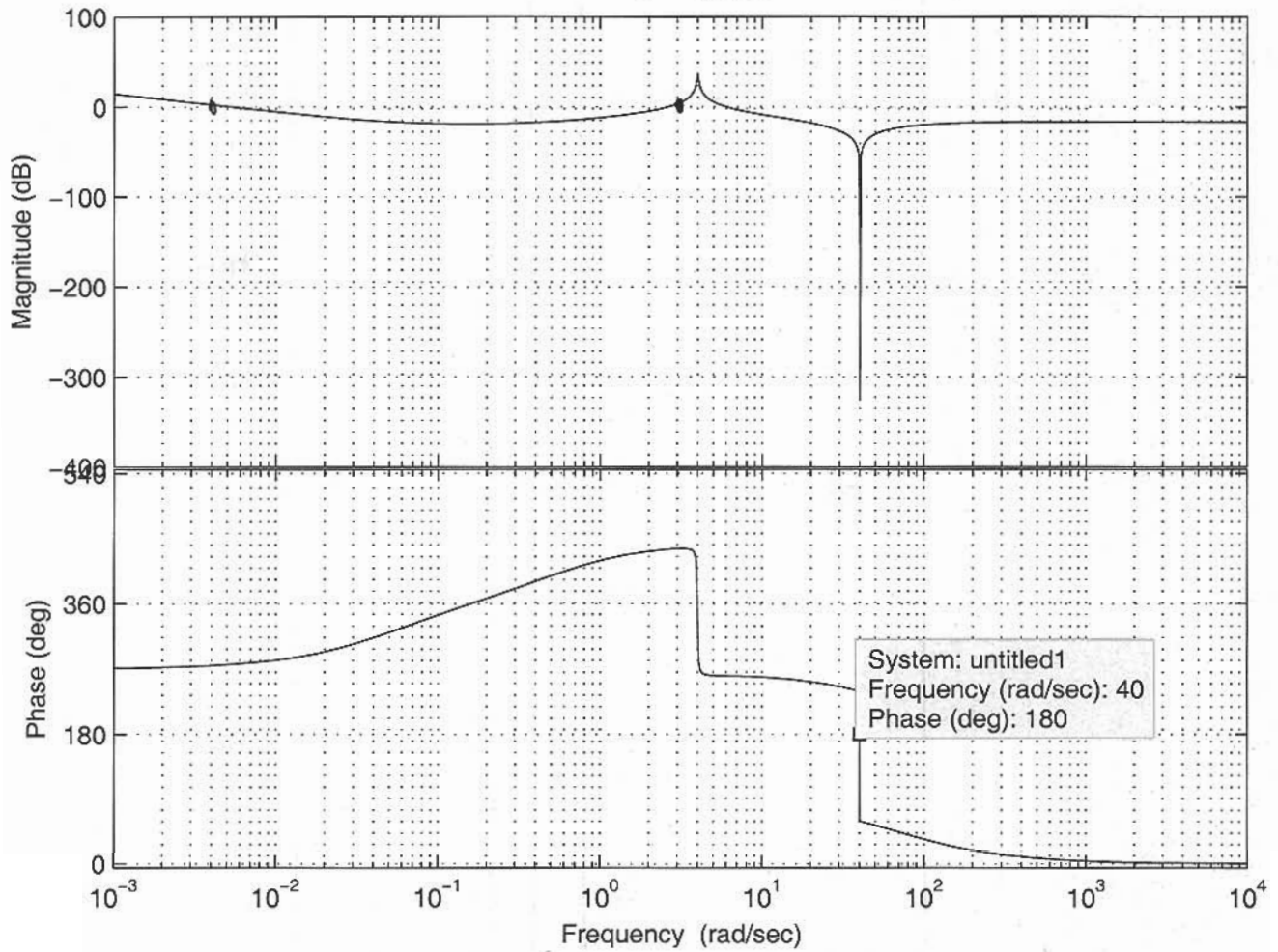
$$K_2 = .0011$$

$$K_1 = .002314$$

v/ controller

#4

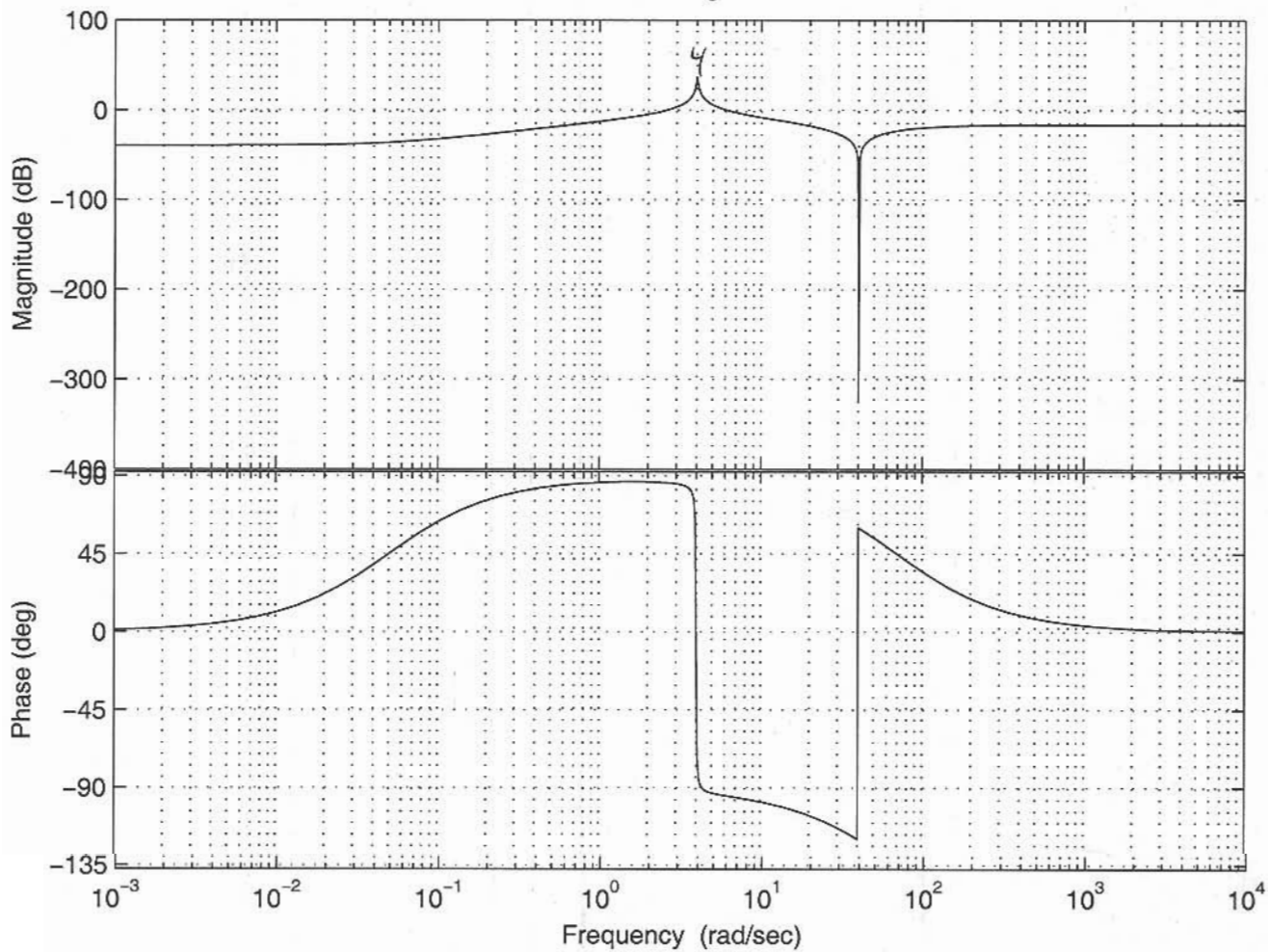
Bode Diagram



w/o controller

#4

Bode Diagram



#5

Need integrator. Need zero to get phase back.
Lead gives better overall response, but not
necessary.

6T

Final solutions

Problem 6.

a) - Assume the velocity is zero - If the external applied force has amplitude less than D , then (say positive), then the switch in the feedback path reacts first with a force of amplitude $-D$ (backwards) to any forward move of the mass: thus the mass cannot move forward - It cannot move backwards either, because as soon as it tries and do that, the reaction force in the ~~total~~ feedback path becomes $+D$ and stops the mass from trying to move backwards -

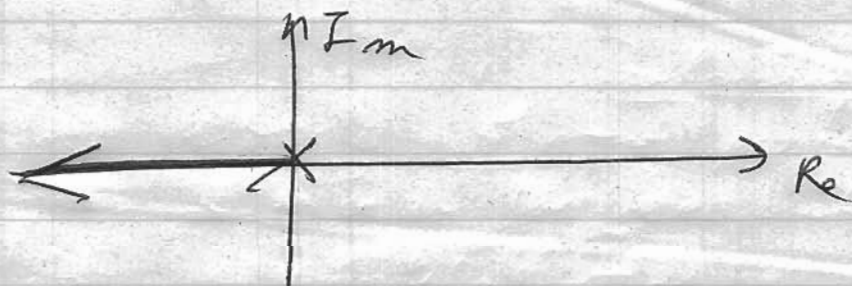
- If the velocity is now strictly positive, then the reaction force in the feedback path is negative, of amplitude $-D$ as intuition suggests.

And vice versa if the velocity is strictly negative -

(6)

- We can use describing function to see, indeed, if there are uncontrolled oscillations - This is not a rigorous argument, but it's OK to use it since not much else is available to you -

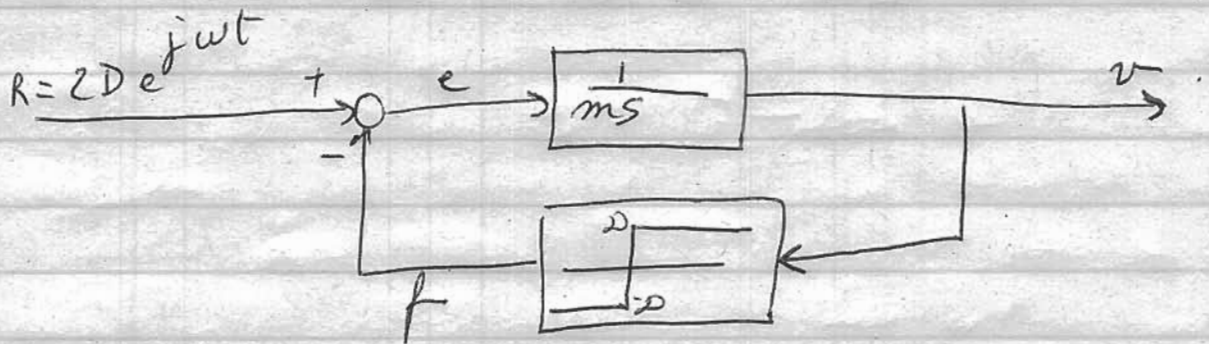
As we have seen, in the case of a switch nonlinearity, there is a limit cycle if the root locus crosses the $j\omega$ axis away from $z=0$ - In one case, the root locus is;



So it never crosses the $j\omega$ axis -
→ there is no limit cycle.

(b) The system is subject to a forcing sinusoidal function of amplitude $2D$

The steady-state ^{velocity} response should be a periodic function.



(c) Let us assume that the only signals of interest are sinusoids -

we have: $e = 2D e^{j\omega t} - f$

we approximate f by: $f = f_0 e^{j(\omega t + \phi_1)}$
 v by: $v = v_0 e^{j(\omega t + \phi_2)}$

and $f_0 = \frac{4D}{\pi v_0} v_0$, $\phi_1 = \phi_2$

and $v = \frac{1}{2j m \omega} e$ or $v_0 = \frac{1}{m \omega} |e|$

Thus ~~$e = 2D e^{j\omega t} - \frac{4D}{\pi v_0} v_0 e^{j(\omega t + \phi_2)}$~~ and: $\phi_1 = \phi_2 = -90^\circ$

Thus:

$$f_{in} = \frac{4D \sin \omega t}{\pi \sin \omega t} \cdot \frac{1}{j m \omega} = \frac{4D}{\pi j}$$

so: ~~$e = 2D \sin \omega t$~~

$$e = \left(2D - \frac{4D}{\pi j} \right) e^{j \omega t}$$

$$\frac{or: e}{2D} = \left(1 - \frac{2}{\pi j} \right) e^{j \omega t}$$

~~So in general the DF for any function is: from R to e: $(1 - \frac{2}{\pi j})$~~

So at the output, we have a "sinusoid" of amplitude $\left| \frac{1}{m j \omega} \left(1 - \frac{2}{\pi j} \right) \right| 2D$.

(e)

(c) in general, we have:

$$e = R e^{j\omega t} - f$$

$$\text{with } f = \frac{4D}{\pi j} e^{j\omega t}$$

so: e and $v = \frac{1}{mj\omega} e$

so: $v = \frac{1}{mj\omega} \left(R - \frac{4D}{\pi j} \right) e^{j\omega t}$

so: $\frac{v}{R e^{j\omega t}} = \frac{1}{mj\omega R} \left(R - \frac{4D}{\pi j} \right)$

Thus: $N(A, \omega) = \left(\frac{1}{m\omega} \right) \left(-j + \frac{4D}{\pi R} \right)$

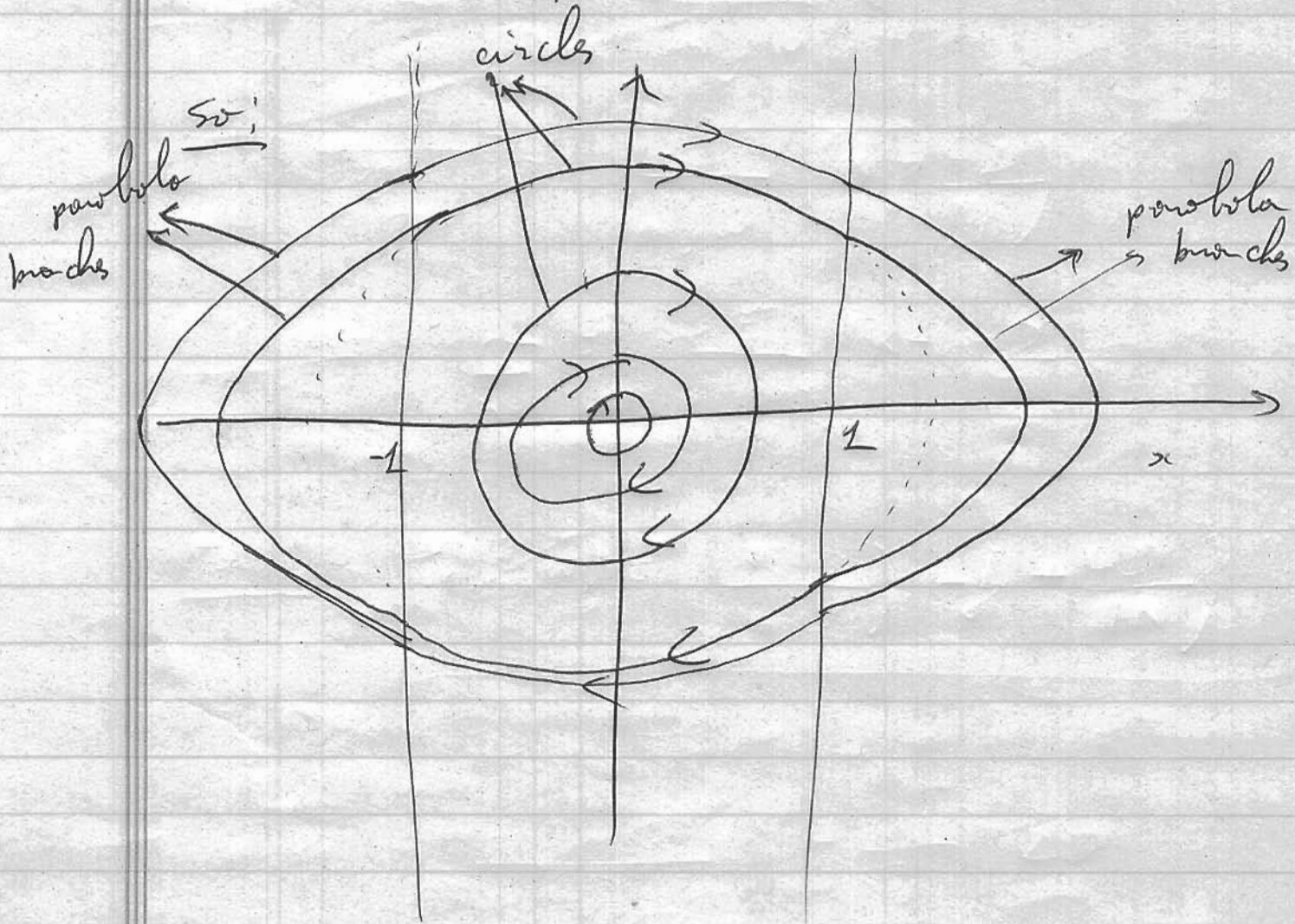
Problem 7

a) The equation of motion is:

$$\ddot{x} = -x \quad \text{for } |x| < 1 \rightarrow \text{circle}$$

$$\ddot{x} = -1 \quad \text{for } x > 1 \rightarrow \text{parabola}$$

$$\ddot{x} = 1 \quad \text{for } x < -1 \rightarrow \text{parabola}$$



b) $\Omega = 0.1 \text{ rad/sec}$

the centrifugal acceleration is
 $\times 0.01 \text{ m/sec}^2$

~~is~~ and is equal to 1 when $x = 100 \text{ m}$.

The equations of motion are:

$\ddot{x} = -0.99x$ for $|x| < 1 \rightarrow$ circle.

$\ddot{x} = -1 + \del{0.01} 0.01x$ for $x > 1$

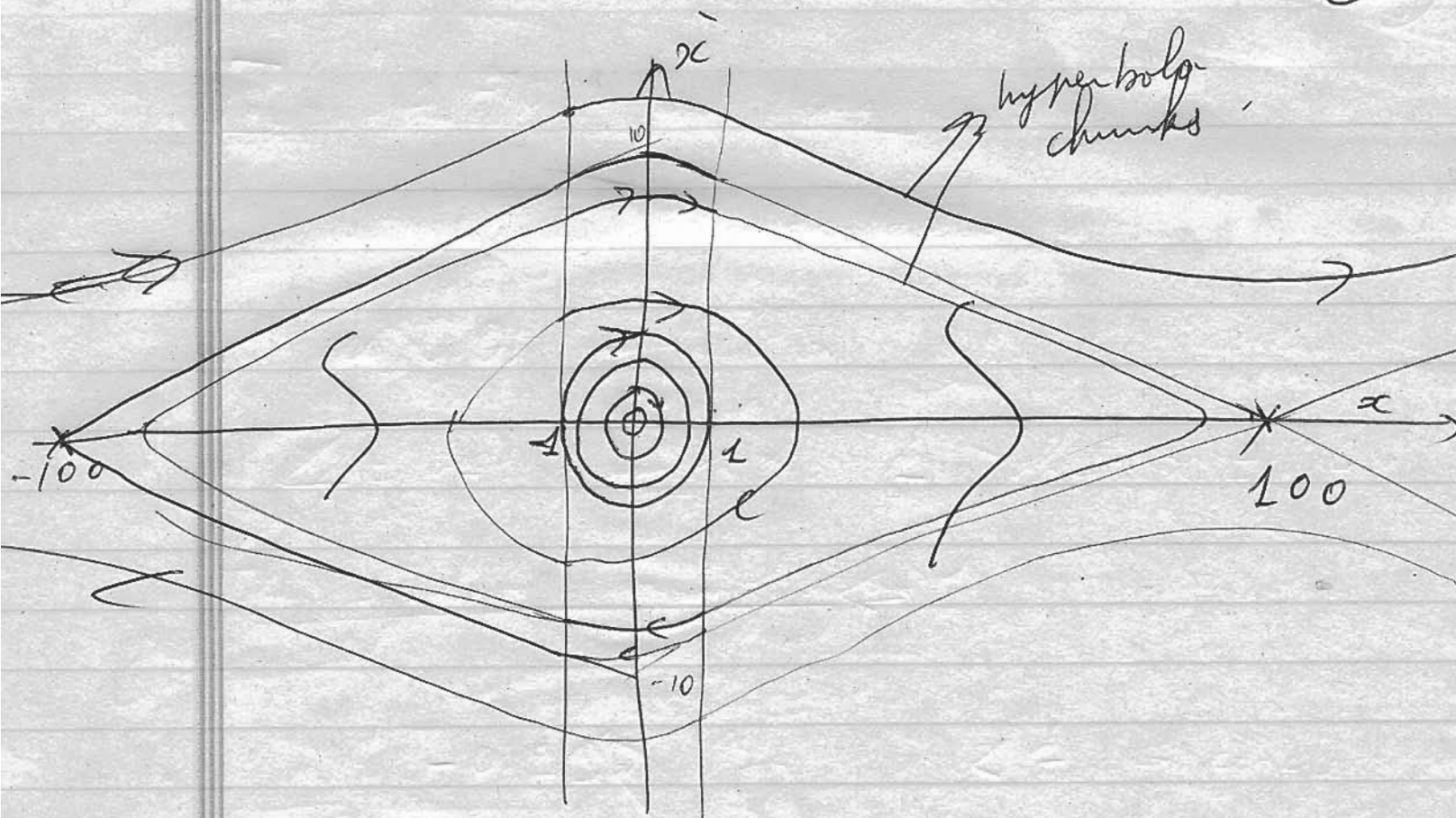
$\ddot{x} = 1 + 0.01x$ for $x < -1$.

what's this?

Analyze! $\ddot{x} = -1 + 0.01x$ in more detail -

we have an equilibrium at $\ddot{x} = 0, x = 100$.
and this is the equation of an inverted pendulum.

(h)



c) assume now Ω spins up!
 The generic equations of motion are!

~~$$\ddot{x} = (-1 + \Omega^2)x$$~~

$$\ddot{x} = (-1 + \Omega^2)x \quad \text{for } |x| < 1$$

$$\ddot{x} = -1 + \Omega^2 x \quad \text{for } x > 1$$

$$\ddot{x} = 1 + \Omega^2 x \quad \text{for } x < -1$$

assume: $\Omega^2 < 1$.

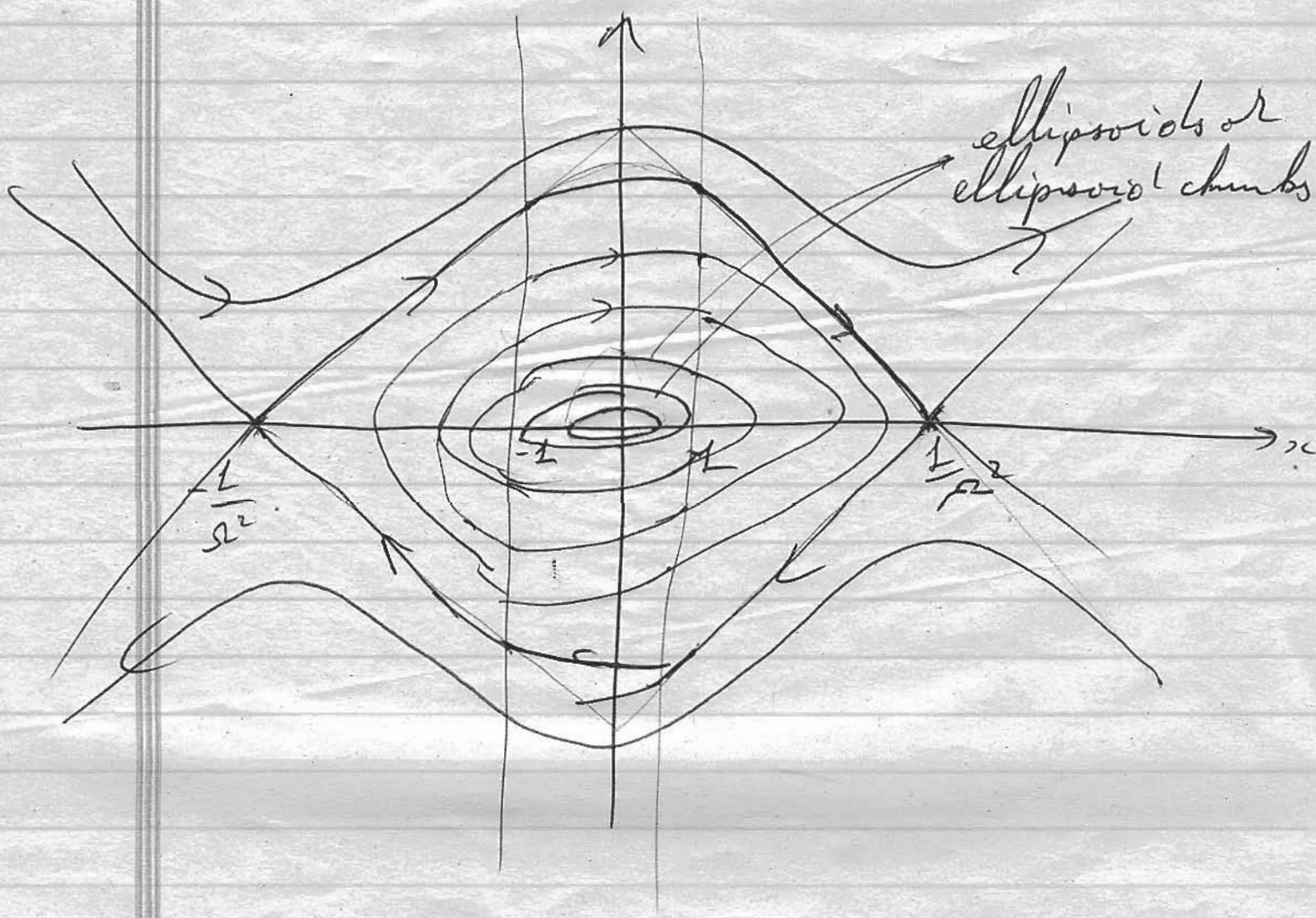
Then! $\ddot{x} = (-1 + \Omega^2)x$ is stable for $|x| < 1$

and! $\ddot{x} = -1 + \Omega^2 x$

has equilibrium at $x = 0$

$$x = \frac{1}{\Omega^2}$$

The phase plane looks like:



(j)

Assume now: $\Omega^2 > 1$.

Then $\ddot{x} = (-1 + \Omega^2)x$ is unstable too!

The phase plane now looks like:

